

CE 2111
Mechanics of Materials-I
(Shear Force and Bending Moment)

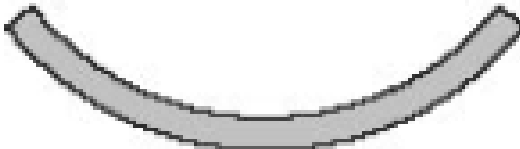
Shear Force and Bending Moment

- The algebraic sum of the vertical forces at any section of a beam to the right or left of the section is known as shear force
- A shear force diagram (SFD) is one which shows the variation of shear force along the length of the beam.
- The algebraic sum of the moments of all the forces acting to the right or left of any section of a beam is known as bending moment
- A bending moment diagram (BMD) is one which shows the variation of bending moment along the length of the beam.

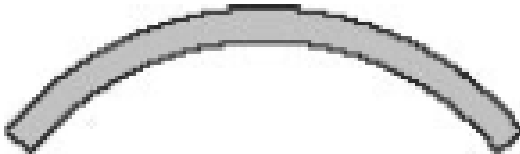
Guidelines for drawing SFD and BMD

- Consider left or right portion of the section
- Add the forces normal to the beam on one of the portion. If right portion of the section is chosen, a force on the right portion acting downward is positive while a force acting upward is downward
- The positive values of shear force and bending moments are plotted above the base line and negative values below the base line
- The shear force diagram increase or decrease suddenly i.e. by vertical straight line at a section where there is a vertical point load
- The shear force between any two vertical loads will be constant and hence the shear force diagram between two vertical loads will be horizontal
- Bending moment at any hinge or roller support, at the end of beam, is zero

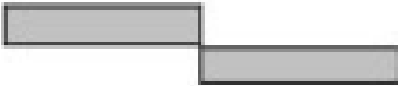
SIGN CONVENTIONS



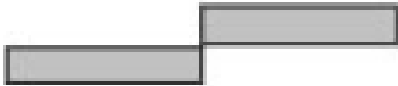
Positive Bending



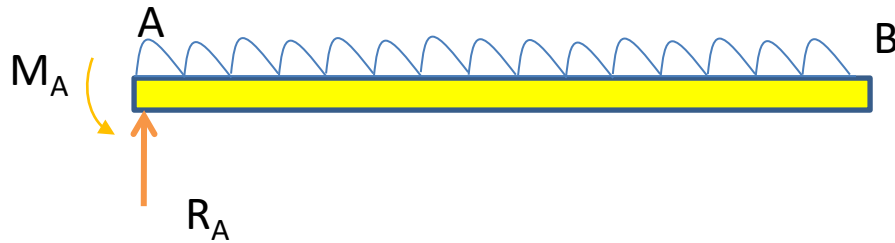
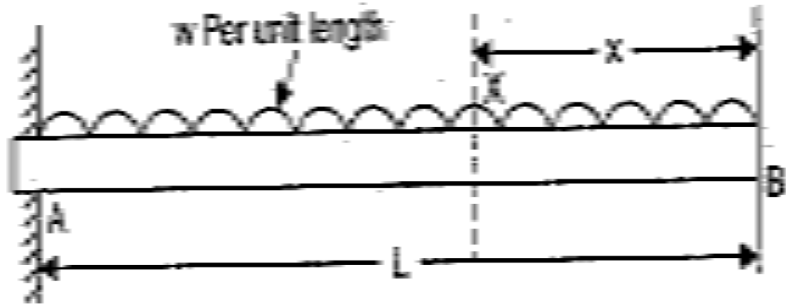
Negative Bending



Positive Shear



Negative Shear

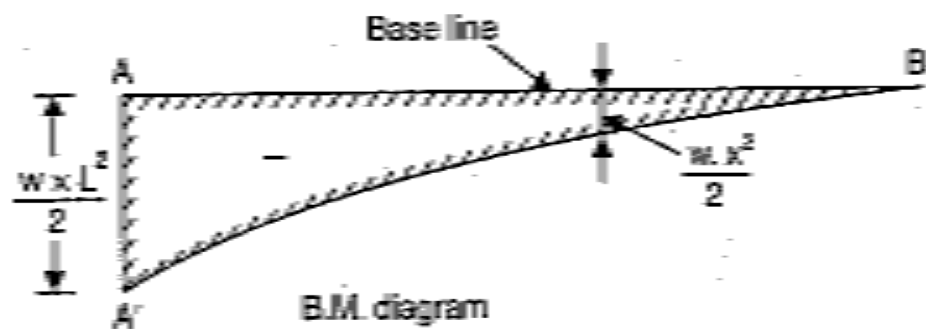
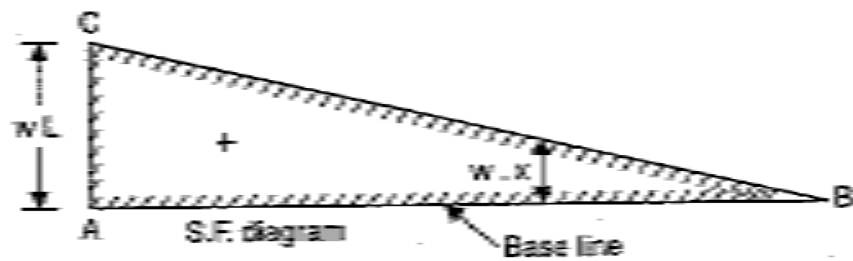
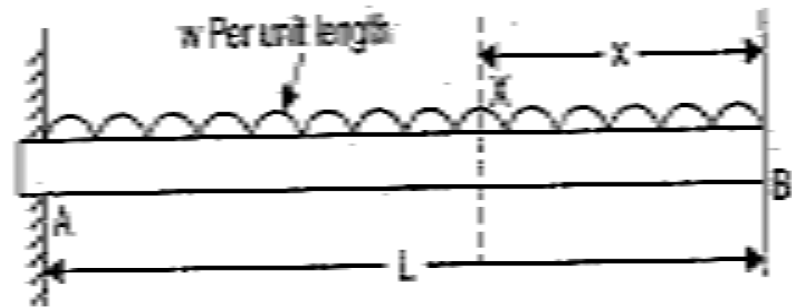


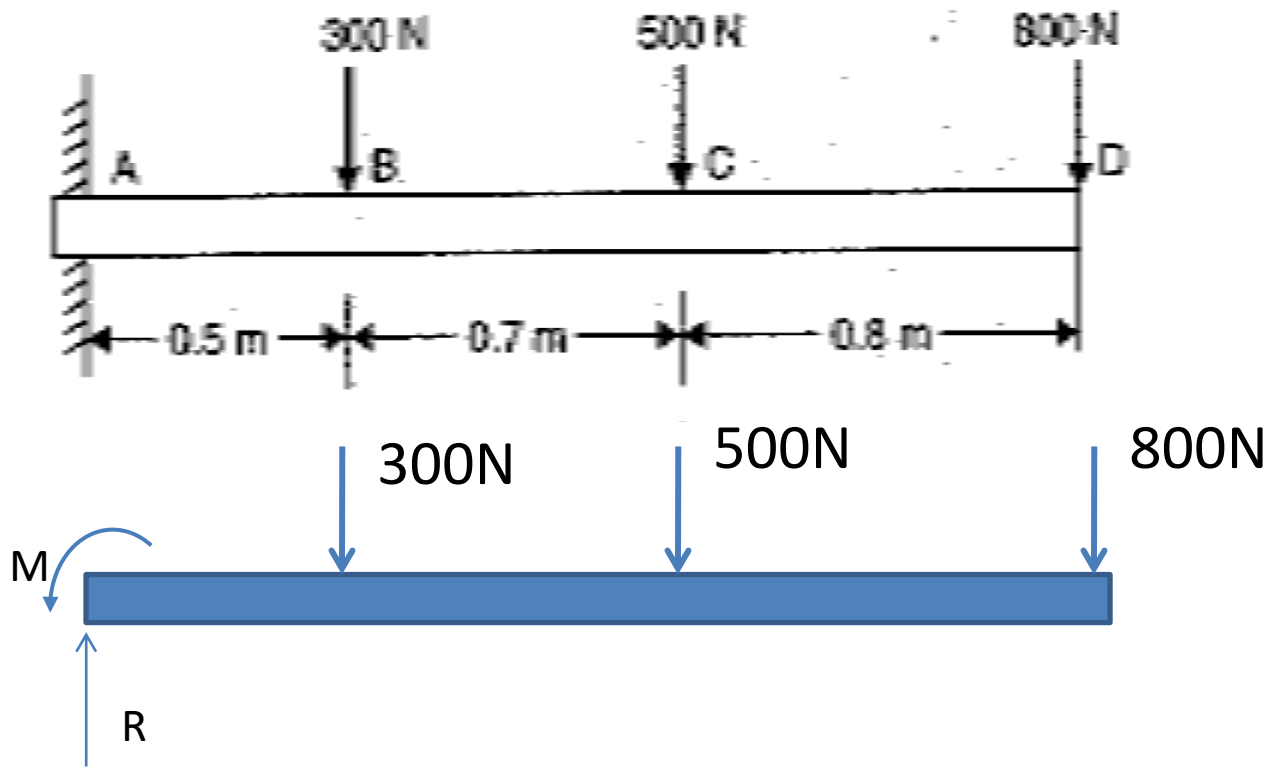
$$R_A = wL$$

$$M_A = wL^2/2$$

Shear force at any section x from right end $V_x = wx$

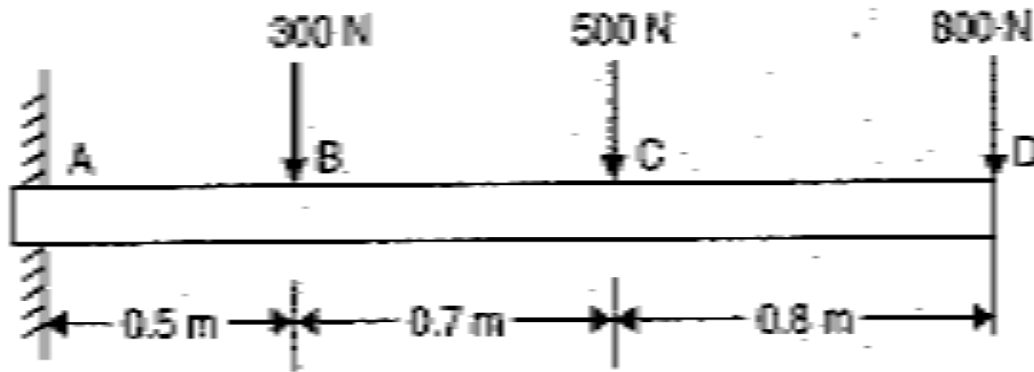
Bending moment at any section x from right end $M_x = wx \cdot x/2 = wx^2/2$



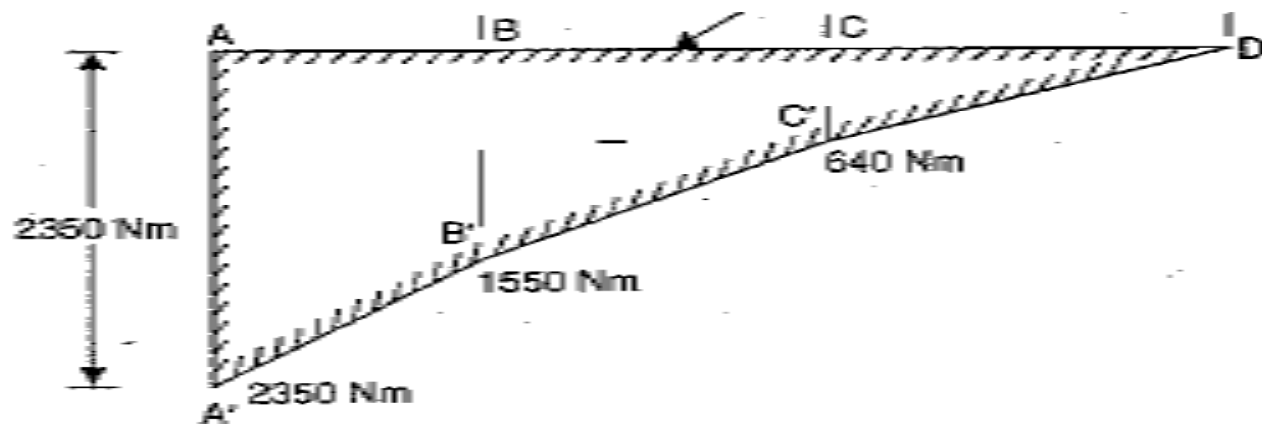
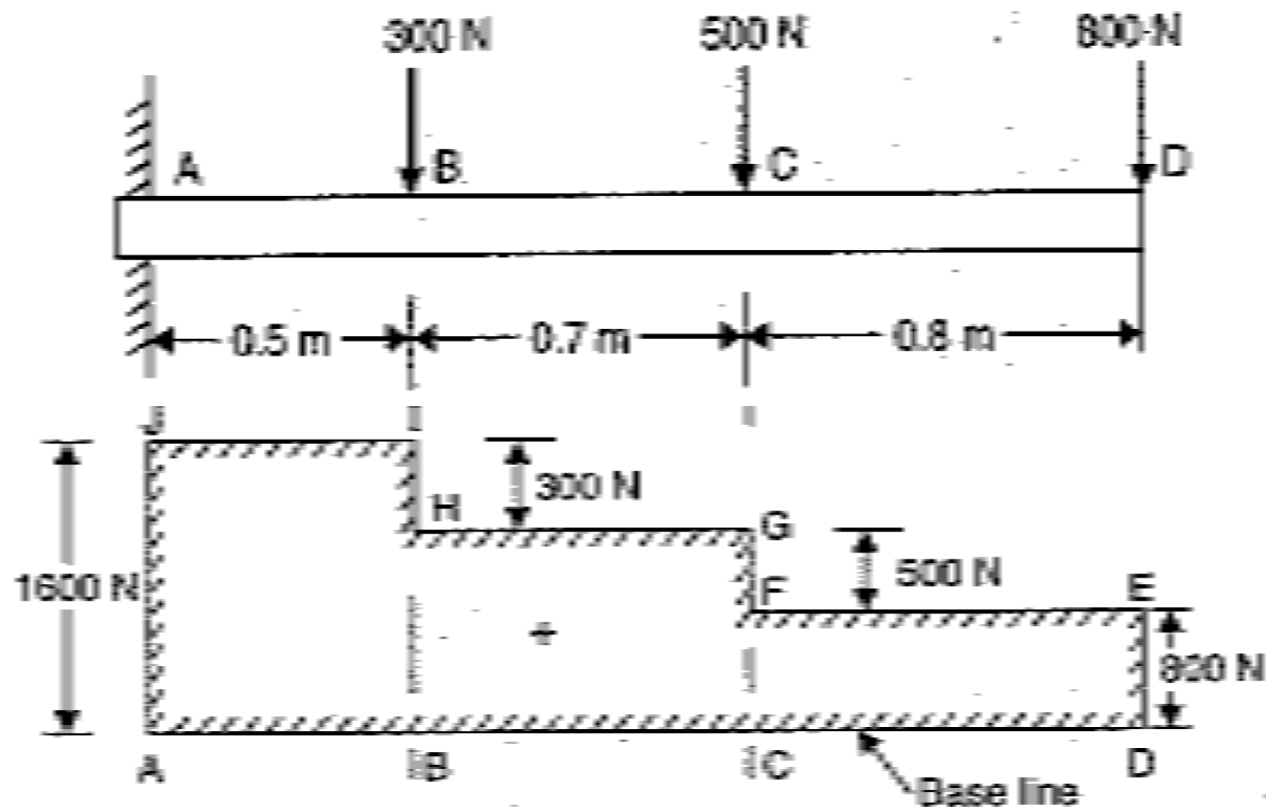


$$R = 1600 \text{ N}$$

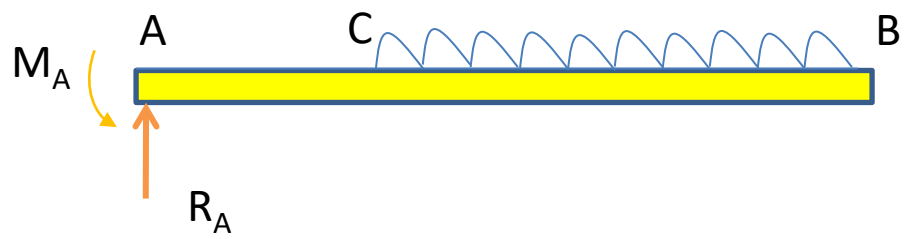
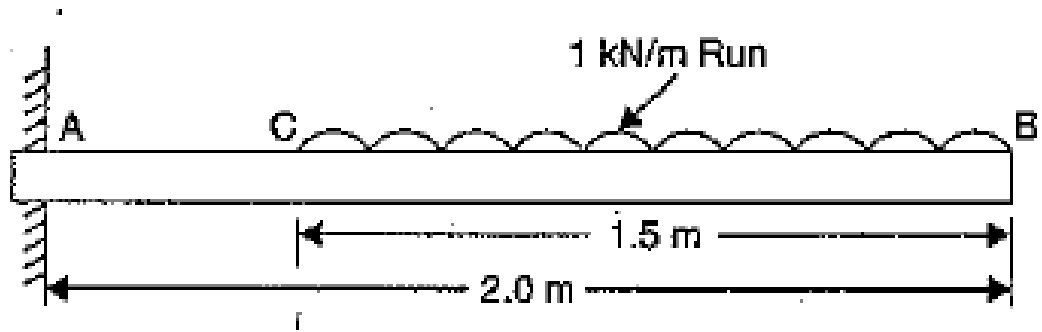
$$M = 2350 \text{ N-m}$$



- Shear force at D $V_D = 800\text{N}$
- Shear force at C $V_C = 800\text{N} + 500\text{N} = 1300\text{N}$
- Shear force at B $V_B = 800\text{N} + 500\text{N} + 300\text{N} = 1600\text{N}$
- Shear force at A $V_A = +1600\text{N}$
- Bending moment at the free end D is zero
- Bending moment between D and C at a distance x from D
 $M_x = -800 * x$
- At C, $x = 0.8\text{m}$ $M_C = -800 * 0.8 = 640\text{N}\cdot\text{m}$
- Bending moment between B and C at a distance x from D
 $M_x = -800 * x - 500(x - 0.8)$
- Bending moment between B and A at a distance x from D
 $M_x = -800 * x - 500(x - 0.8) - 300(x - 1.5)$

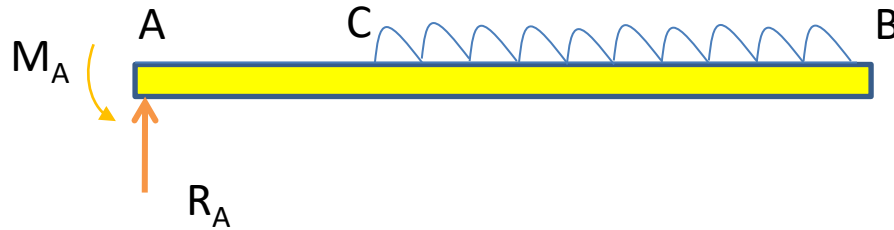


Draw SFD and BMD for the Cantilever Beam

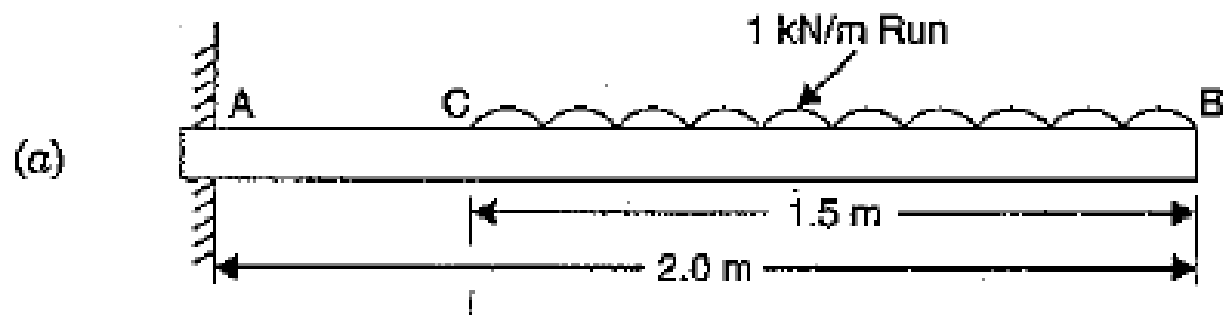


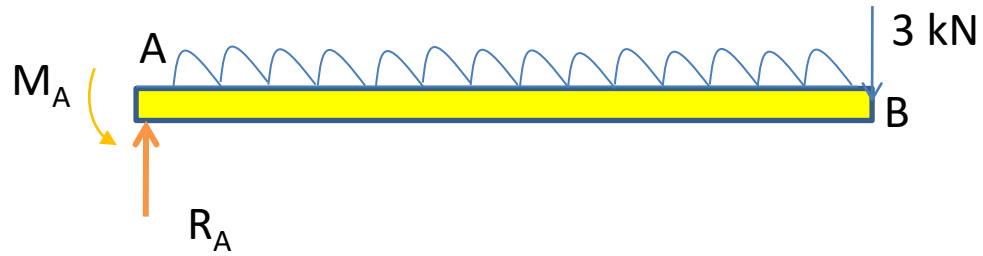
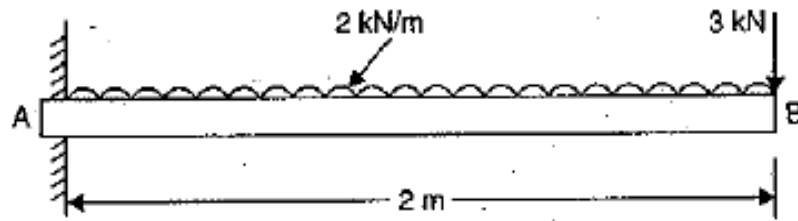
$$R_A = 1.5 \text{ kN}$$

$$M_A = 1.875 \text{ kN-m}$$



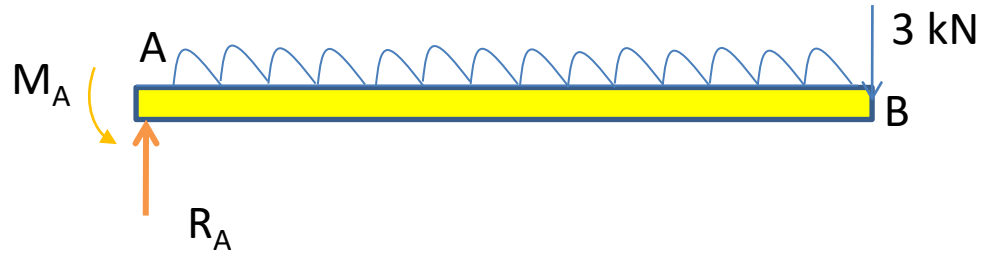
- Consider any section between C and B $V_x = 1.0 * x$ kN
- At B; $x = 0$ $V_B = 0$
- At C; $x = 1.5$ $V_C = 1.5$ kN
- Between A and C there is no load, the shear will remain constant
- Bending moment at any section between C and B,
 $M_x = -1.0 * x * x / 2 = -x^2 / 2$
- At B $x = 0$ $M_B = 0$
- At C, $x = 1.5$ $M_C = -1.125$ N-m
- Bending moment between A and C, $M_x = -(\text{Load due to UDL}) * (x - 0.75)$ [$x = 1.5 - 2.0$ m]



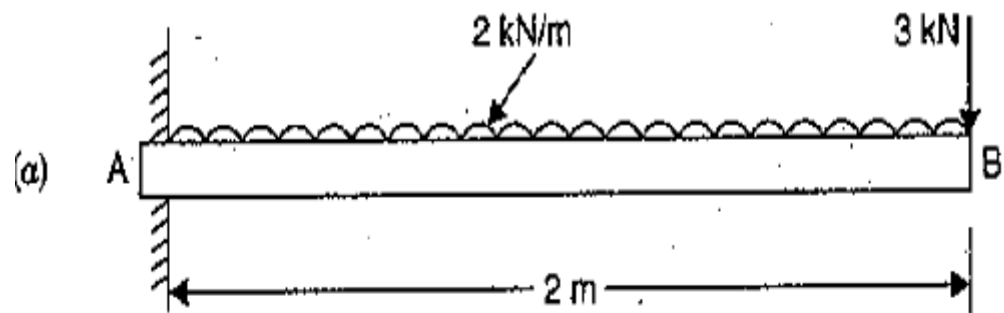


$$R_A = 7 \text{ kN}$$

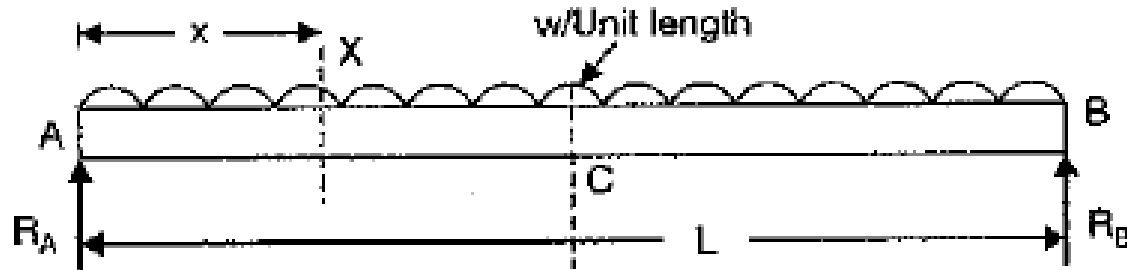
$$M_A = 10 \text{ kN-m}$$



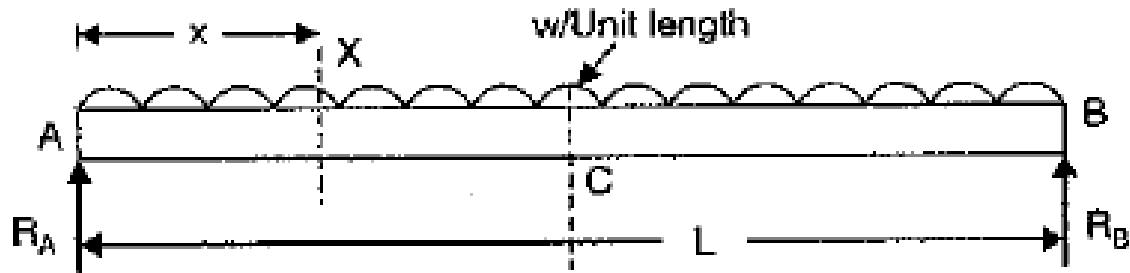
- Shear force at B $V_B = 3.0 \text{ kN}$
- Shear force at any section x from free end
- $V_x = 3.0 + w \cdot x = 3.0 + 2 \cdot x$
- At B $x = 0 \quad V_B = 3.0 \text{ N}$
- At A $x = 2.0 \quad V_A = 3.0 + 2 \cdot 2 = 7 \text{ N}$
- Bending moment at any section from free end
- $M_x = -(3 \cdot x + 2 \cdot x \cdot x / 2) = -(3x + x^2)$
- At B $x = 0 \quad M_B = 0$
- At A, $x = 2 \quad M_A = -10 \text{ kN-m}$
- Bending moment between A and C, $M_x = -(\text{Load due to UDL}) \cdot (x - 0.75) \quad [x = 1.5 - 2.0 \text{ m}]$



Draw SFD and BMD for the simple Supported Beam

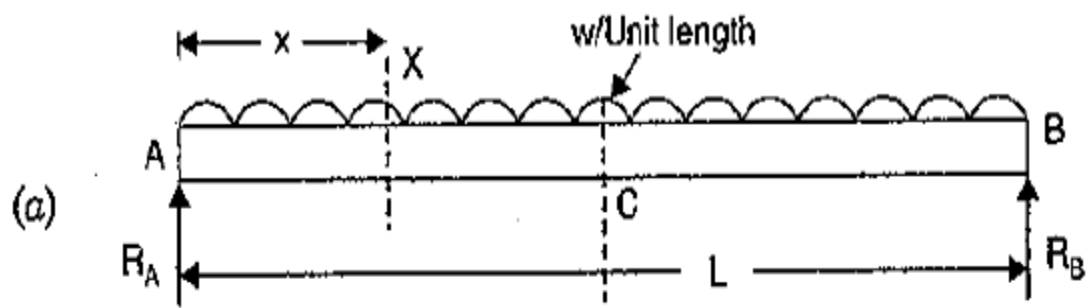


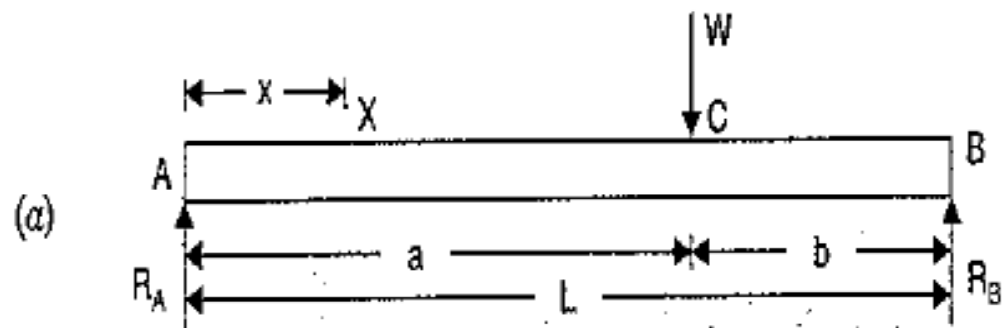
$$R_A = R_B = wL/2$$

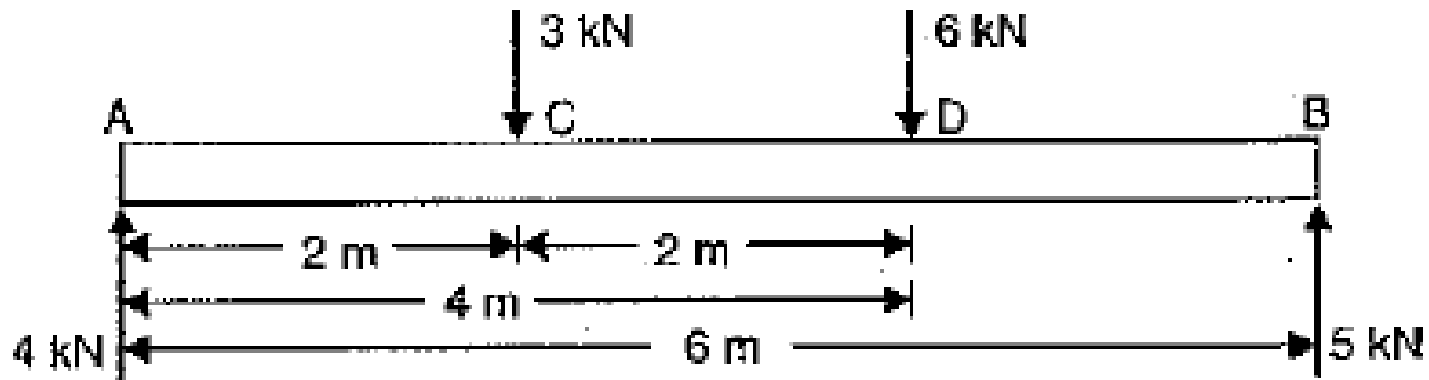


- Consider any section at distance x from left support

$$V_x = R_L - w \cdot x = wL/2 - w \cdot x$$
- At A , $x=0 \Rightarrow V_A = wL/2$
- At B , $x=L \Rightarrow V_B = -wL/2$
- At C , $x = L/2 \Rightarrow V_C = 0$
- Bending moment at any section x from A
- $$M_x = R_L \cdot x - wx^2/2 = w(Lx - x^2)/2$$
- At A , $x=0 \Rightarrow M_A = 0$
- At B , $x=L \Rightarrow M_B = 0$
- At C , $x = L/2 \Rightarrow M_C = wL^2/8$



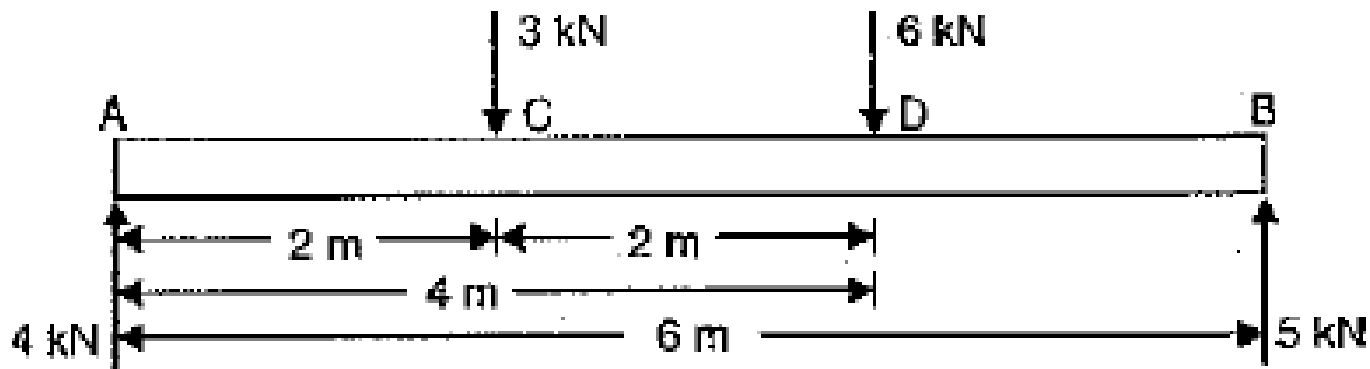




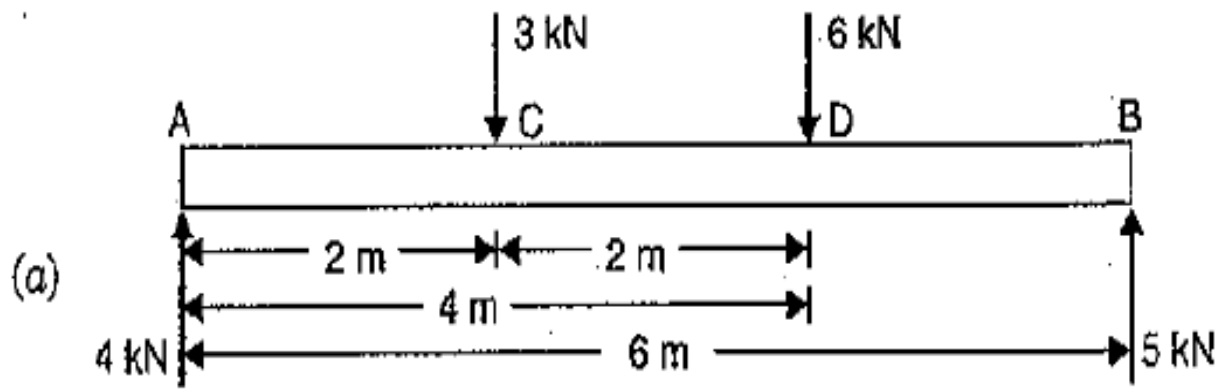
$$\sum M_A = 0; \quad 3kN * 2 + 6kN * 4 - R_B * 6 = 0$$

$$R_B = 5 \text{ kN}$$

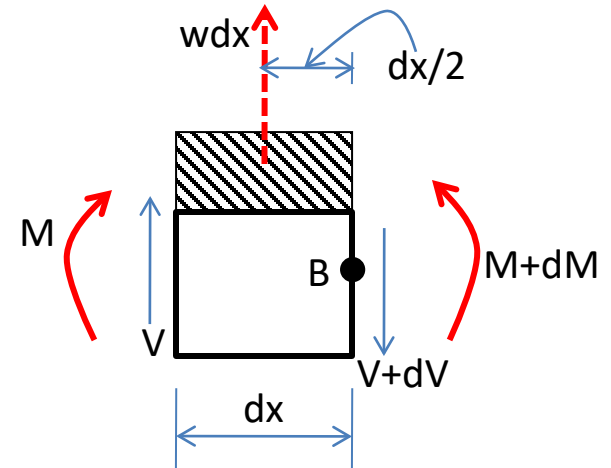
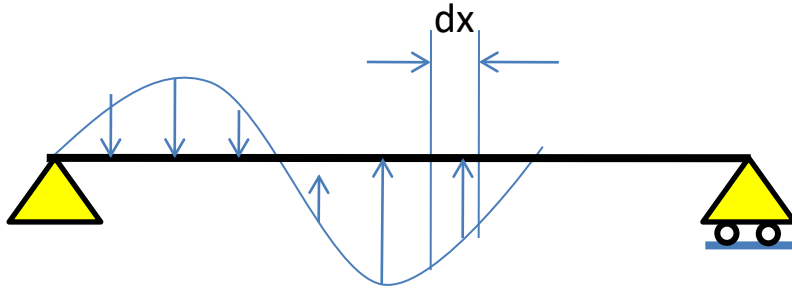
$$R_A = 4 \text{ kN}$$



- Shear force at A $= R_A = 4 \text{ kN}$
- Shear force between A and C is constant and equal to 4 kN
- Shear force at C $V_C = 4 - 3 = 1 \text{ kN}$
- Shear force at D $V_D = 1 - 6 = -5 \text{ kN}$
- Shear force at B $V_B = -5 \text{ kN}$
- **Bending moment at A $M_A = 0$**
- Bending moment at C $M_C = R_A * 2 = 4 * 2 = 8 \text{ kN-m}$
- Bending moment at D $M_D = R_A * 4 - 3 * 2 = 10 \text{ kN-m}$
- Bending moment at B $M_B = 0$



Relations among load, shear and moment



$$\sum F_y = V + w dx - (V+dV) = 0$$

$$dV = w dx$$

$$\int_{V_1}^{V_2} dV = \int_{x_1}^{x_2} w dx$$

$$V_2 - V_1 = \Delta V = (\text{Area})_{\text{load}}$$

$$\sum M_B = M + V dx + (w dx) dx/2 - (M + dM) = 0$$

$$dM = V dx$$

$$\int_{M_1}^{M_2} dM = \int_{x_1}^{x_2} V dx$$

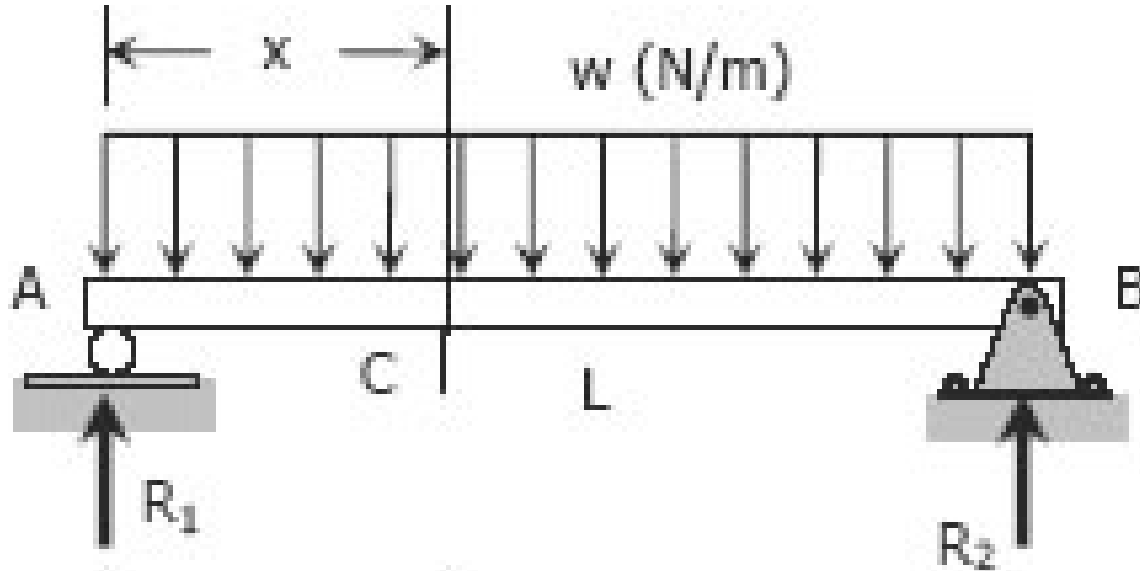
$$M_2 - M_1 = \Delta M = (\text{Area})_{\text{shear}}$$

$$w = \frac{dV}{dx} = \text{slope of shear diagram}$$

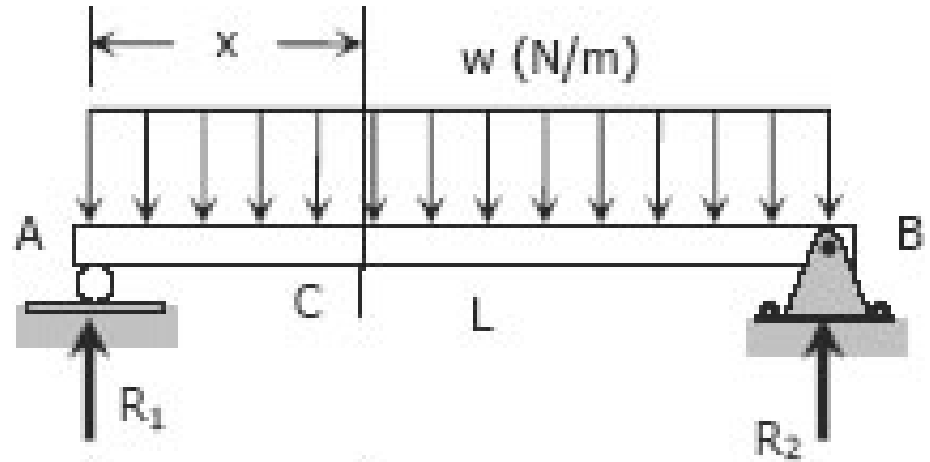
$$V = \frac{dM}{dx} = \text{slope of moment diagram}$$

Relation among Load Shear and Moment

Consider a simple beam shown of length L that carries a uniform load of w (N/m) throughout its length and is held in equilibrium by reactions R_1 and R_2 .



$$R_1 = R_2 = wL/2$$



- $V_x = R_1 - wx = wL/2 - wx$
- $M_x = R_1 * x - w * x^2/2$
- $M_x = wLx/2 - w * x^2/2$
- Differentiate M with respect to x

$$\frac{dM}{dx} = \frac{wL}{2} \frac{dx}{dx} - \frac{w}{2} 2x \frac{dx}{dx} \quad \Rightarrow \quad \frac{dM}{dx} = \frac{wL}{2} - wx$$

$$\therefore \frac{dM}{dx} = V$$

- Thus, the rate of change of the bending moment with respect to x is equal to the shearing force, or **the slope of the moment diagram at the given point is the shear at that point.**

- Differentiating V with respect to x

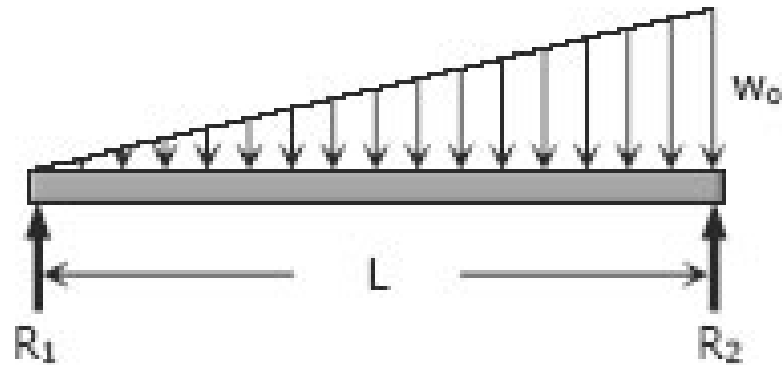
$$\frac{dV}{dx} = -w = \text{Load}$$

Thus, the rate of change of the shearing force with respect to x is equal to the load or **the slope of the shear diagram at a given point equals the load at that point.**

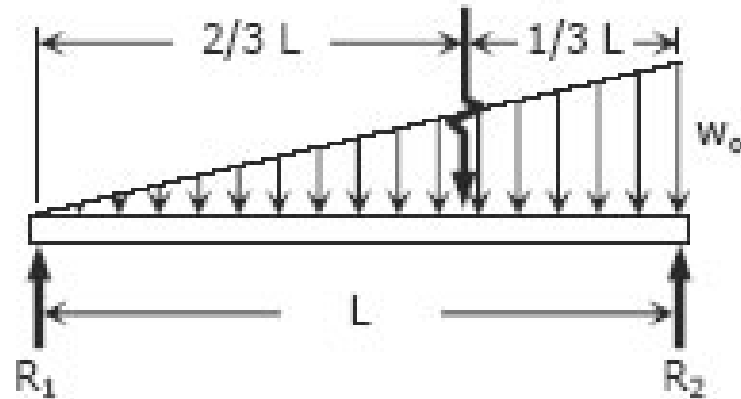
PROPERTIES OF SHEAR AND MOMENT DIAGRAMS

- The following are some important properties of shear and moment diagrams:
- The area of the shear diagram to the left or to the right of the section is equal to the moment at that section.
- The slope of the moment diagram at a given point is the shear at that point.
- The slope of the shear diagram at a given point equals the load at that point.
- The maximum moment occurs at the point of zero shears.
- When the shear diagram is increasing, the moment diagram is concave upward.
- When the shear diagram is decreasing, the moment diagram is concave

Draw SFD and BMD



$$F = \frac{1}{2} L w_0$$

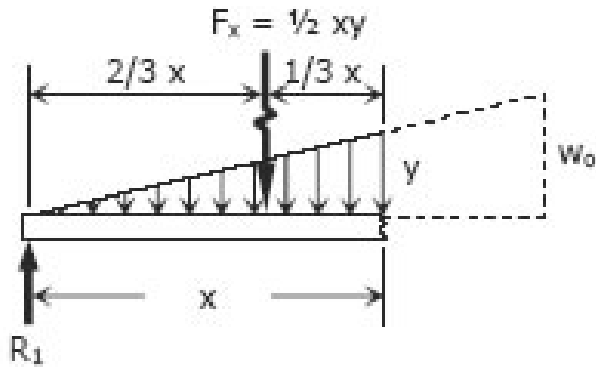


$$\sum M_{R2} = 0$$

$$R_1 = \frac{1}{6} L w_0$$

$$\sum M_{R1} = 0$$

$$R_2 = \frac{1}{3} L w_0$$



$$\frac{y}{x} = \frac{w_0}{L} \quad y = \frac{w_0}{L}x$$

$$F_x = \frac{1}{2}xy = \frac{1}{2}x\left(\frac{w_0}{L}x\right) = \frac{w_0}{2L}x^2$$

$$V = R_1 - F_x = \frac{1}{6}Lw_0 - \frac{w_0}{2L}x^2$$

$$M = R_1x - F_x\left(\frac{1}{3}x\right) = \frac{1}{6}Lw_0x - \frac{w_0}{6L}x^3$$

When $x=0$; $V = 1/6Lw_0 = R_1$

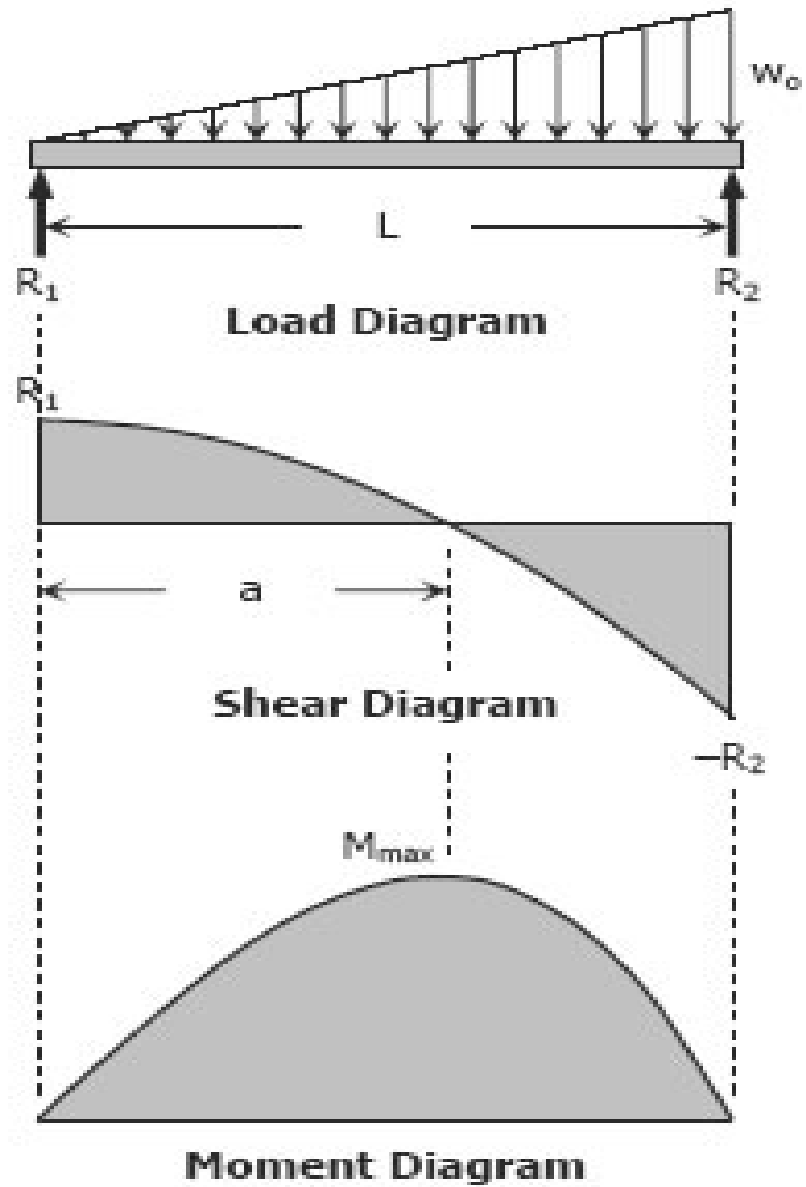
$X=L$; $V = -1/3Lw_0 = -R_2$

Location of zero shear from left end $V=0$ i.e. $a = 0.577L$

When $x=0$; $M=0$

$X=L$; $M=0$

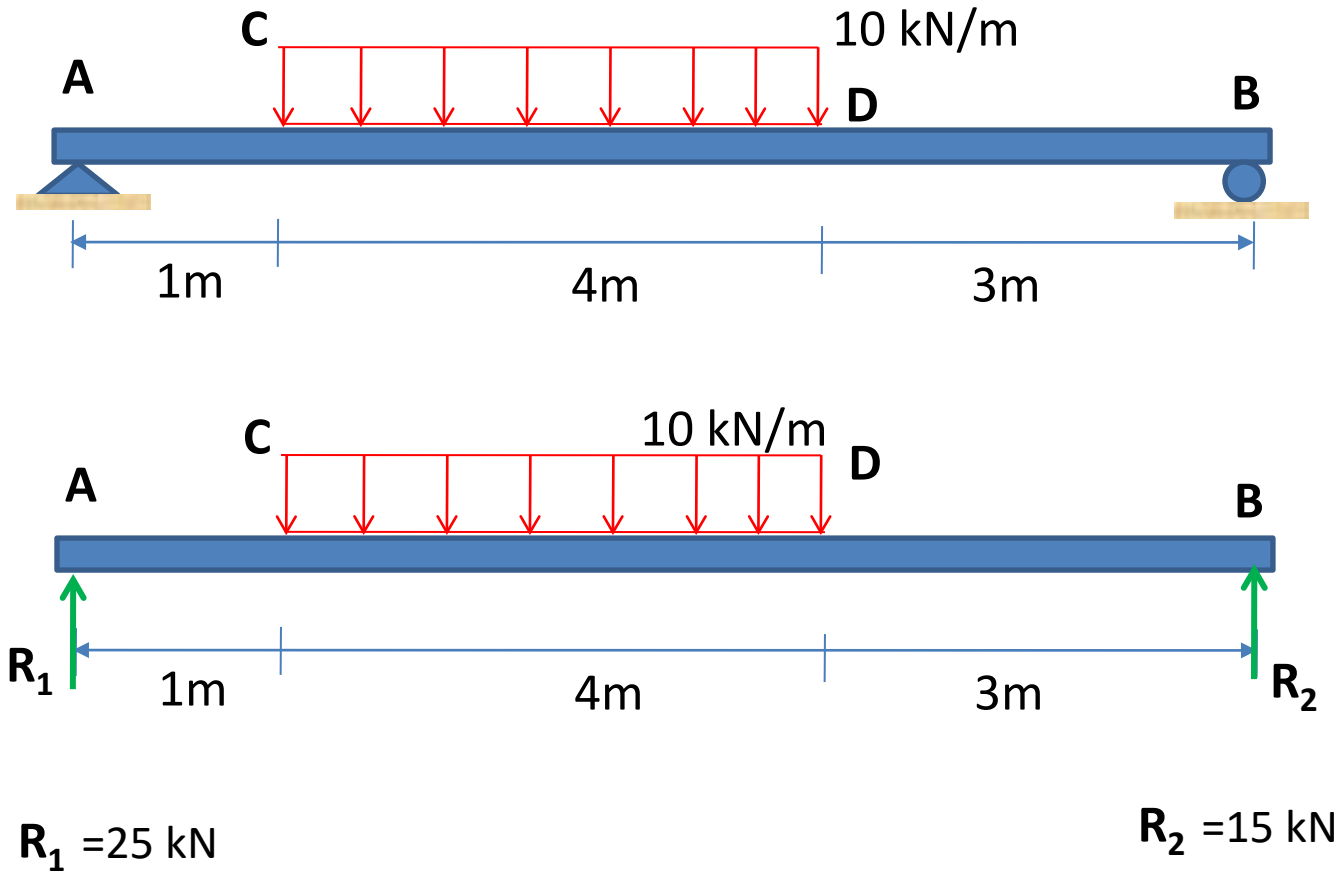
Moment at zero shear $M_{\max} = 0.064L^2w_0$

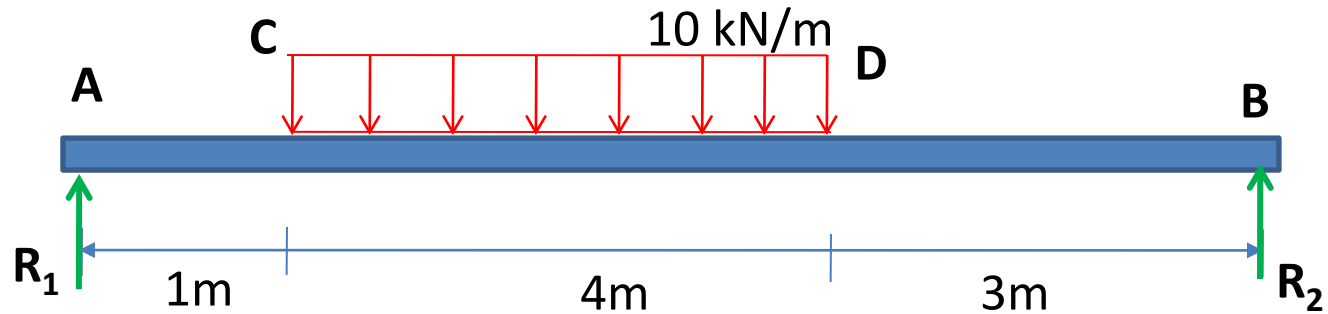


Nature of Load, SF and BM

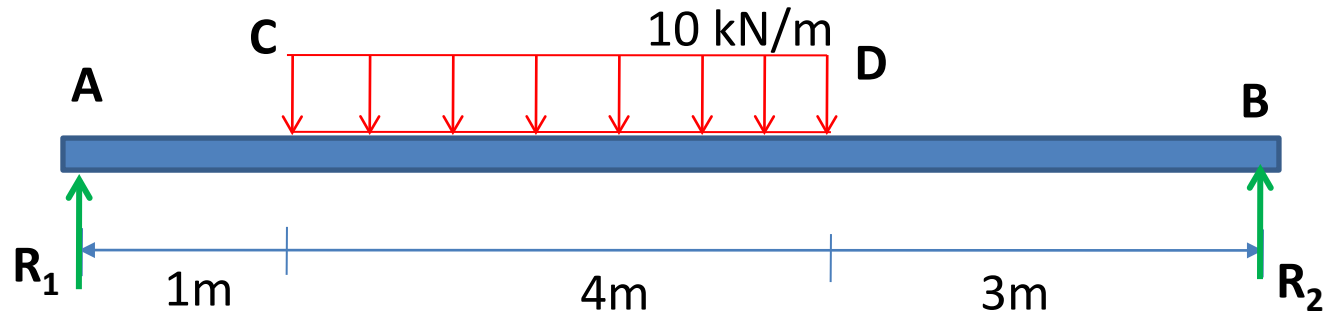
$w(x)$	Concentrated load	UDL	Triangular
$V(x)$	Straight Line, No slope	Sloped Straight Line	Parabolic
$M(x)$	Sloped Straight Line	Parabolic	Cubic

Draw the SFD and BMD for the simple supported beam shown in figure below

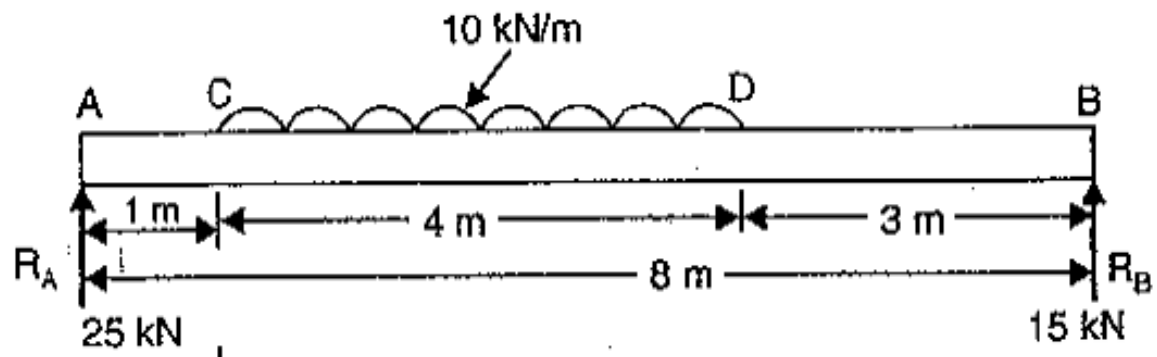


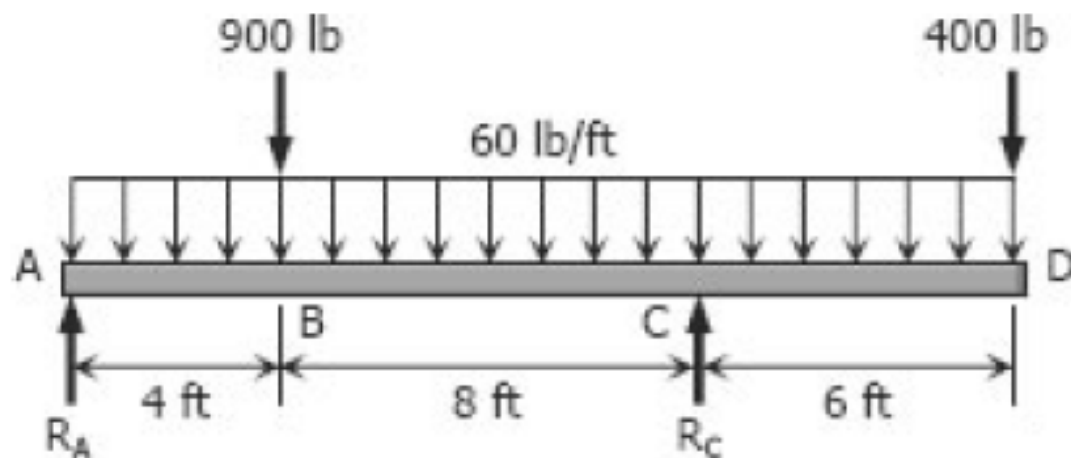


- Shear force at A $V_A = R_A = 25 \text{ kN}$
- Shear force at C $V_C = 25 \text{ kN}$
- Shear force at any section between C and D at a distance x from A
- $V_x = 25 - 10*(x - 1)$
- At D $x = 5$ and $V_D = 25 - 10*(5-1) = -15 \text{ kN}$
- Shear force at B $V_B = -15 \text{ kN}$
- Location of zero shear
- $0 = 25 - 10*(x-1)$
- $x = 35/10 = 3.5 \text{ m from A}$



- Simply supported beam
- Bending moment at A and B is 0
- Bending moment at any section between C and D at a distance x from A
- $M_x = R_A * x - 10 * (x-1) * (x-1) / 2$
- $M_x = 25x - 5 (x-1)^2$
- At C $x = 1$ $M_C = 25$ kN-m
- At D $x = 5$ $M_D = 25 * 5 - 5 * (5-1)^2 = 45$ kN-m
- B.M. at zero shear $M_{\max} = 25 * 3.5 - 5 * (3.5-1)^2 = 56.25$ kN-m





$$\sum M_A = 0$$

$$12R_C = 4(900) + 18(400) + 9[(60)(18)]$$

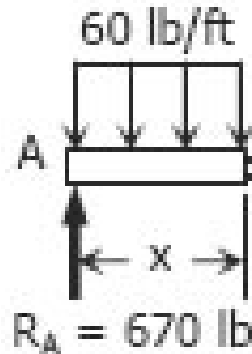
$$R_C = 1710 \text{ lb}$$

$$\sum M_C = 0$$

$$12R_A + 6(400) = 8(900) + 3[60(18)]$$

$$R_A = 670 \text{ lb}$$

Part AB



$$V_{AB} = 670 - 60x \text{ lb}$$

$$M_{AB} = 670x - 60x(x/2)$$
$$= 670x - 30x^2 \text{ lb-ft}$$

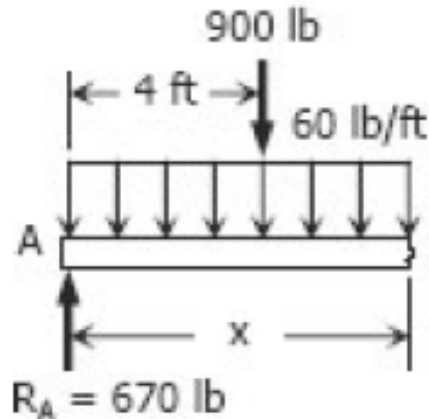
$$\text{At } x = 0 \text{ ft } V_{AB} = 670 - 60 \cdot 0 = 670 \text{ lb}$$

$$\text{At } x = 4 \text{ ft } V_{AB} = 670 - 60 \cdot 4 = 430 \text{ lb}$$

$$\text{At } x = 0 \text{ ft } M_{AB} = 0$$

$$\text{At } x = 4 \text{ ft } V_{AB} = 670 \cdot 4 - 30 \cdot 4^2 = 2200 \text{ lb-ft}$$

Part BC



$$\begin{aligned}V_{BC} &= 670 - 900 - 60x \\ &= -230 - 60x \text{ lb}\end{aligned}$$

$$\begin{aligned}M_{BC} &= 670x - 900(x - 4) - 60x(x/2) \\ &= 3600 - 230x - 30x^2 \text{ lb-ft}\end{aligned}$$

$$\text{At } x = 4 \text{ ft } V_{BC} = -230 - 60 \cdot 4 = -470 \text{ lb}$$

$$\text{At } x = 12 \text{ ft } V_{BC} = -230 - 60 \cdot 12 = -950 \text{ lb}$$

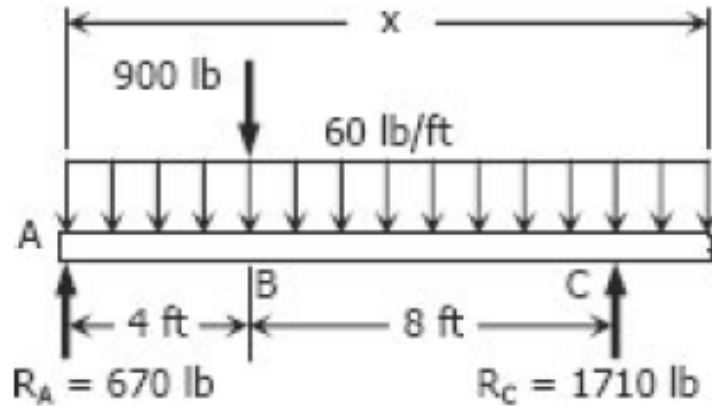
$$\text{At } x = 4 \text{ ft } M_{BC} = 3600 - 230 \cdot 4 - 30 \cdot 4^2 = 2200 \text{ lb-ft}$$

$$\text{At } x = 12 \text{ ft } M_{BC} = 3600 - 230 \cdot 12 - 30 \cdot 12^2 = -3480 \text{ lb-ft}$$

Location of zero bending moment

$$M_{BC} = 0 \text{ or } x = 15.44 \text{ ft and } 7.77 \text{ ft}$$

Part CD



$$V_{CD} = 670 + 1710 - 900 - 60x$$

$$V_{CD} = 1480 - 60x$$

$$\text{At } x = 12\text{ft } V_{CD} = 1480 - 60 * 12 = 760 \text{ lb}$$

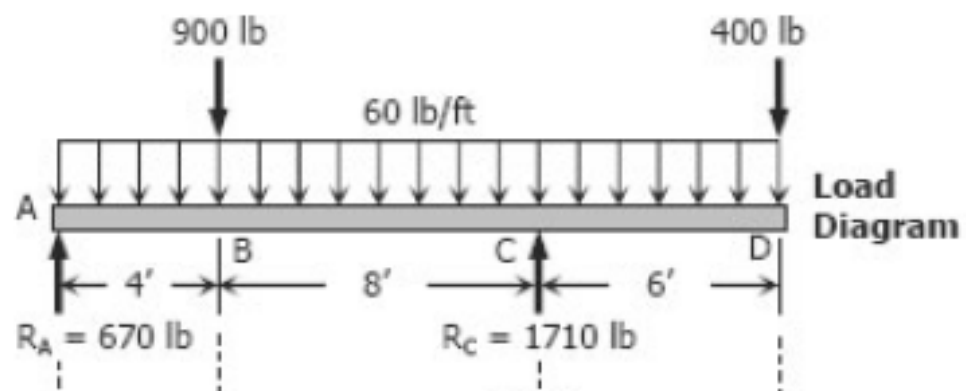
$$\text{At } x = 18 \text{ft } V_{CD} = 1480 - 60 * 18 = 400 \text{ lb}$$

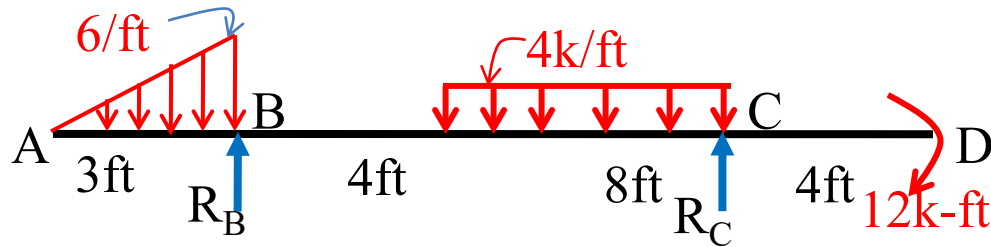
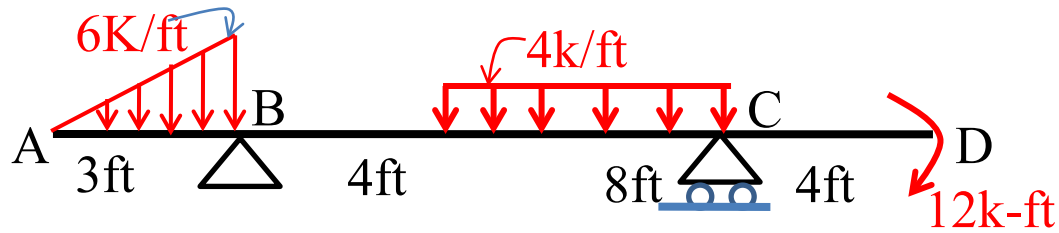
$$M_{CD} = 670x + 1710(x-12) - 900(x-4) - 60x(x/2)$$

$$M_{CD} = -16920 + 1480x - 30x^2$$

$$\text{At } x = 12\text{ft } M_{CD} = -16920 + 1480*12 - 30*12^2 = -3480 \text{ lb-ft}$$

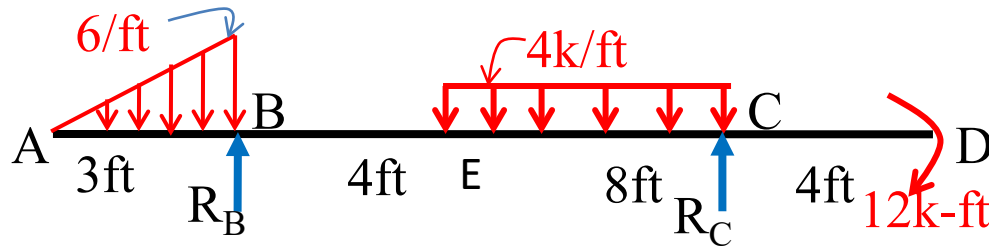
$$\text{At } x = 18\text{ft } M_{CD} = -16920 + 1480*18 - 30*18^2 = 0 \text{ lb-ft}$$





$$\sum M_B = -\frac{1}{2} * 3 * 6 * (\frac{1}{3} * 3) + 4 * 8(4+4) + 12 - R_C * 12 = 0$$

$$R_B = 19.42k, \quad R_C = 21.58K$$



$$4 \frac{x^2}{2} - 21.58x + 12 = 0$$

$$X = 0.59 \text{ ft}$$

$$R_B = 19.42 \text{ k}, \quad R_C = 21.58 \text{ K}$$

Shear at D $V_D = 0$

Shear at C $V_C = -R_C = -21.58 \text{ K}$

Shear at E $V_E = -21.58 + 4 \cdot 8 = 10.42 \text{ K}$

Shear at B $V_B = -21.58 + 4 \cdot 8 - 19.42 = -9 \text{ K}$

Shear at A = 0

Location of zero shear between C and E

$$-21.58 + 4 \cdot x = 0; \quad x = 5.4 \text{ ft from C}$$

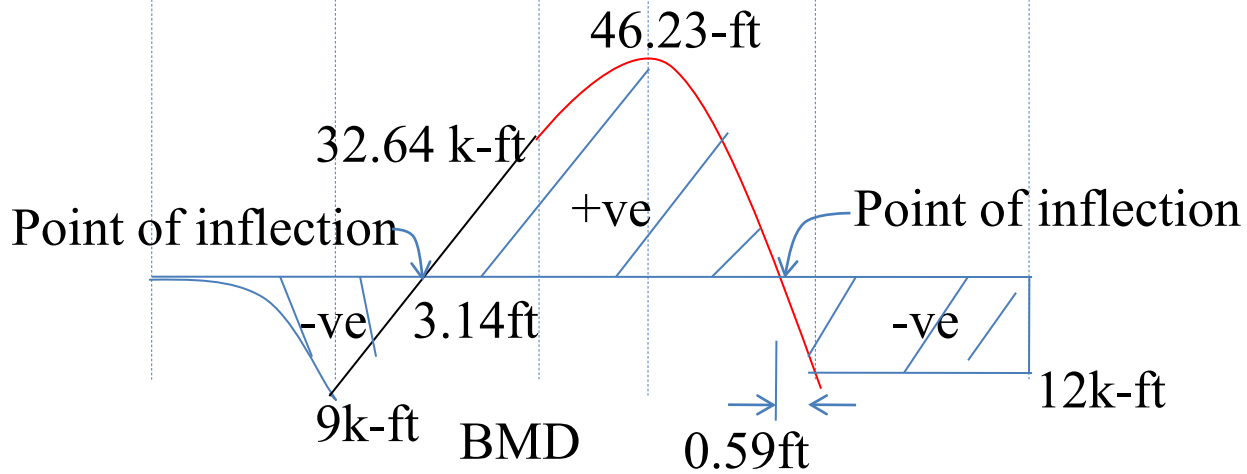
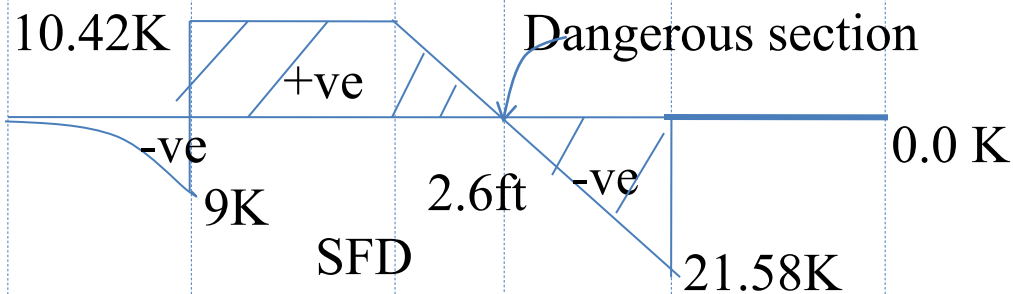
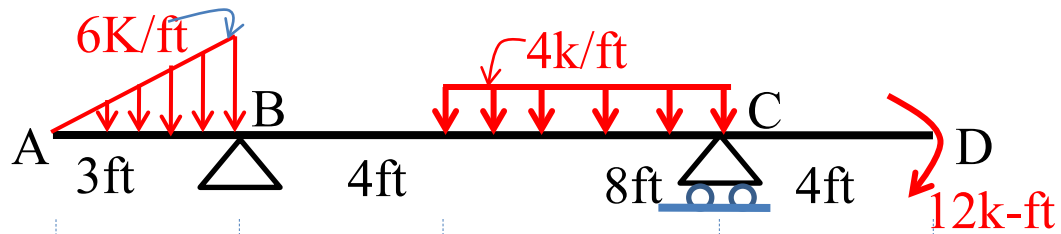
Bending moment at D $M_D = -12 \text{ k-ft}$

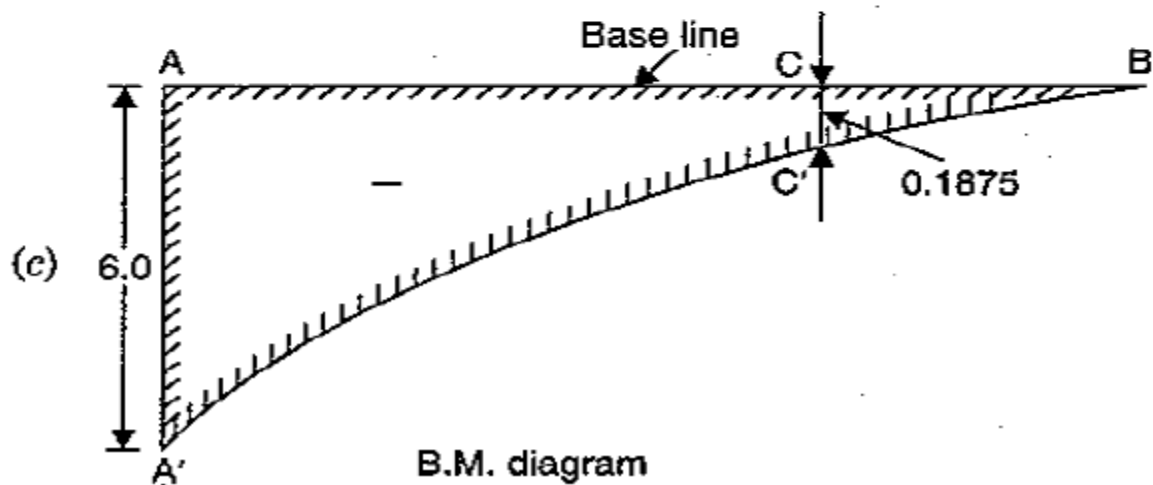
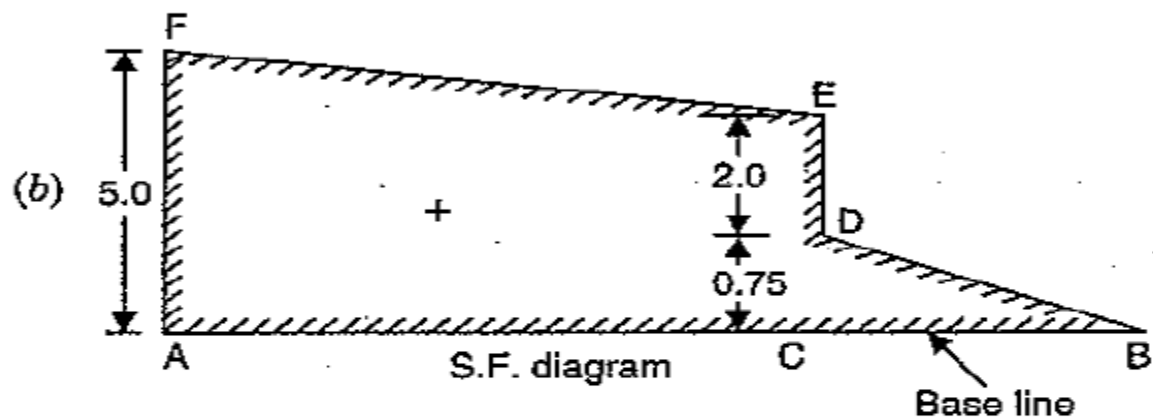
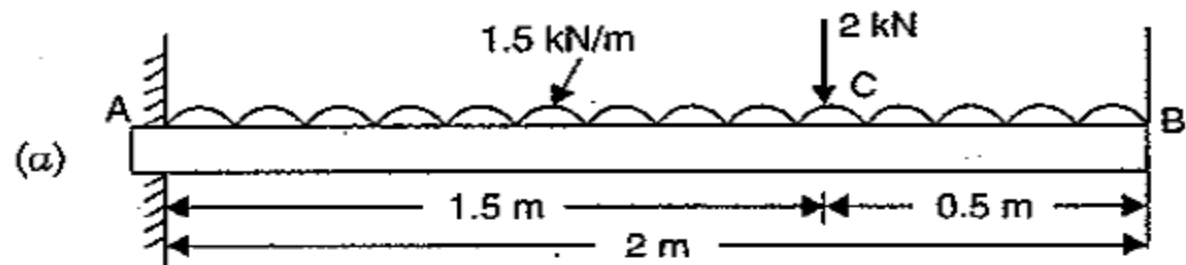
Bending moment at C $M_C = -12 \text{ k-ft}$

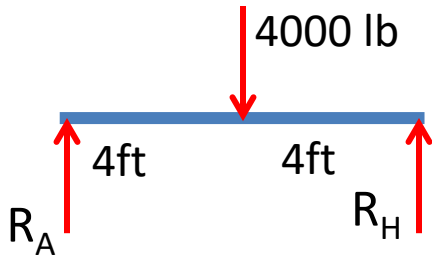
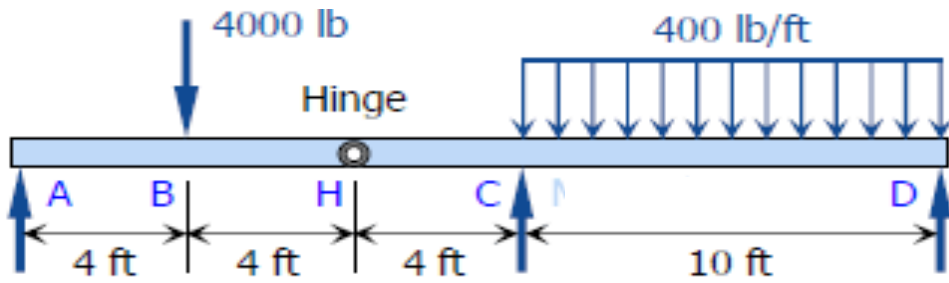
Bending moment at E $M_E = -12 + (21.58 \cdot 8) - 4 \cdot 8 \cdot 8 / 2 = 32.64 \text{ k-ft}$

Bending moment at B $M_B = -12 + (21.58 \cdot 12) - 4 \cdot 8 \cdot (4 + 8/2) = -9 \text{ k-ft}$

Maximum moment $M_{\max} = -12 + 21.58 \cdot 5.4 - 4 \cdot 5.4 \cdot 5.4 / 2 = 46.2 \text{ k-ft}$

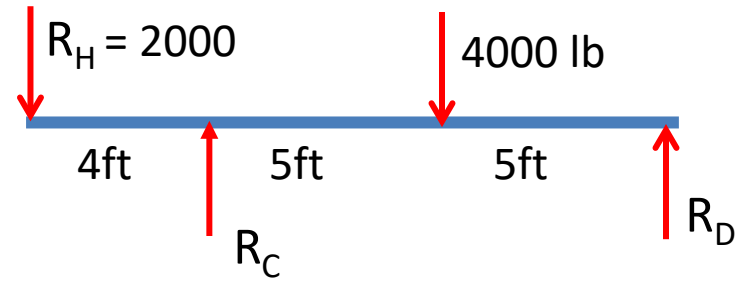






$$\sum M_A = 0; 4000 * 4 - R_H * 8 = 0$$

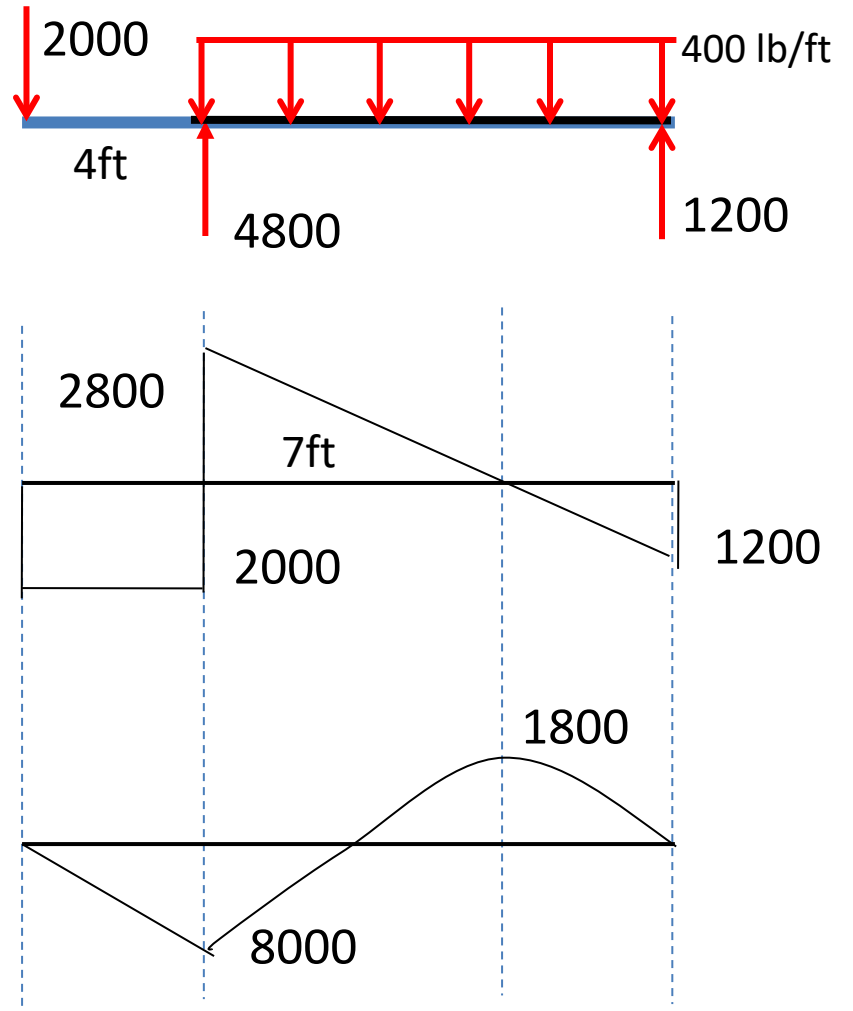
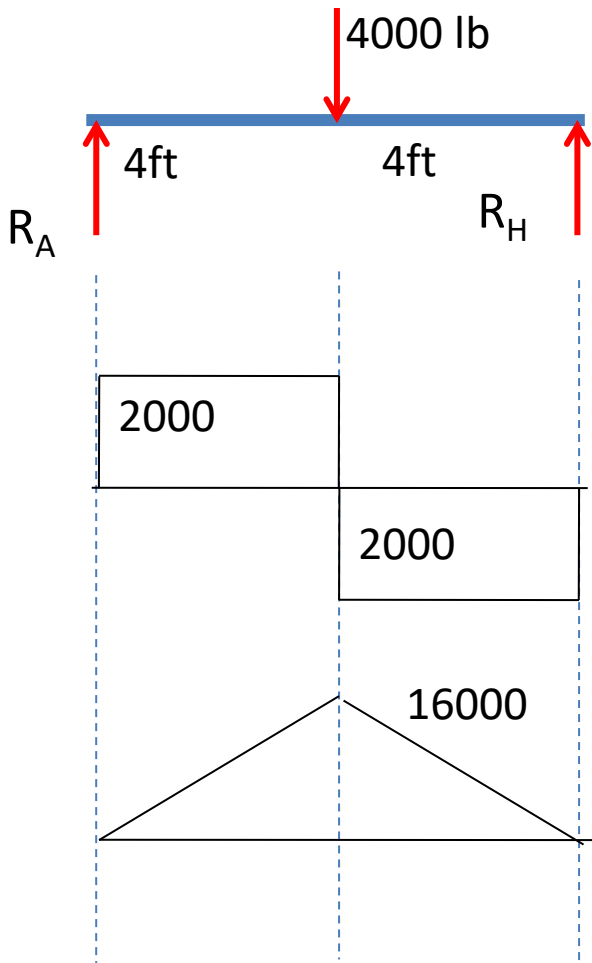
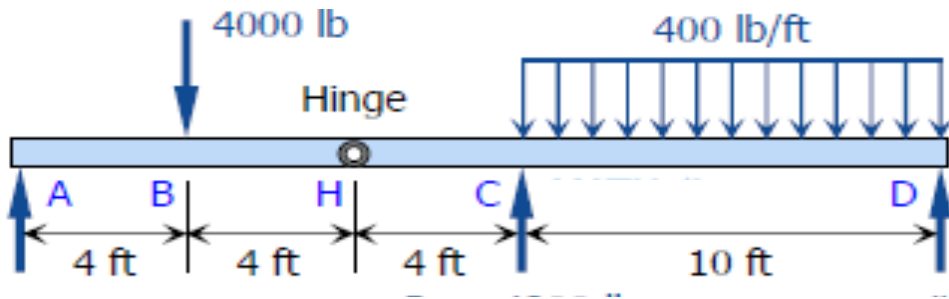
$$R_H = 2000 \text{ lb}$$

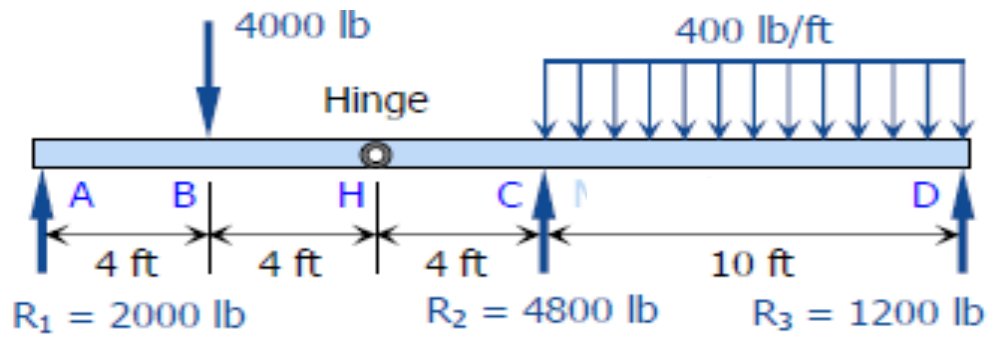


$$\sum M_D = 0; R_C * 10 - 4000 * 5 - 2000 * 14 = 0$$

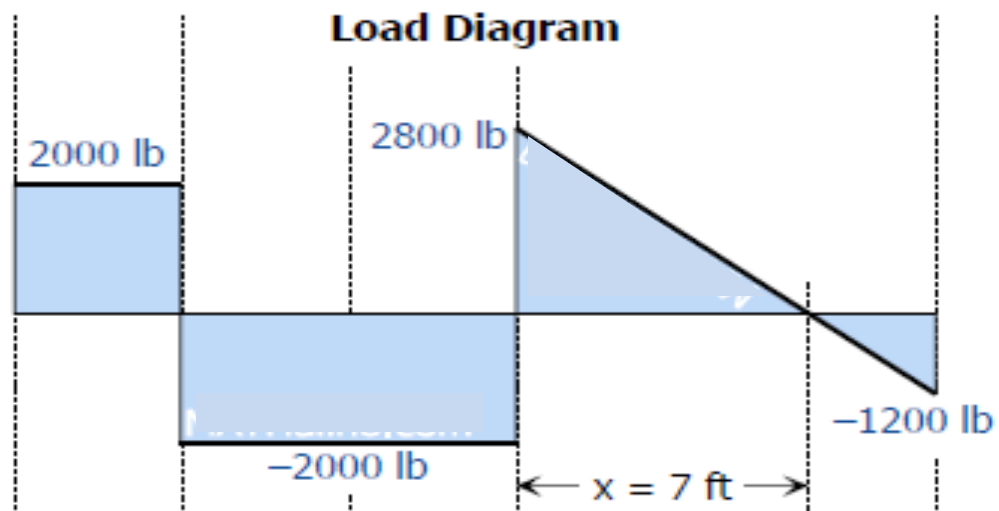
$$R_C = 4800 \text{ lb}$$

$$R_D = 1200 \text{ lb}$$

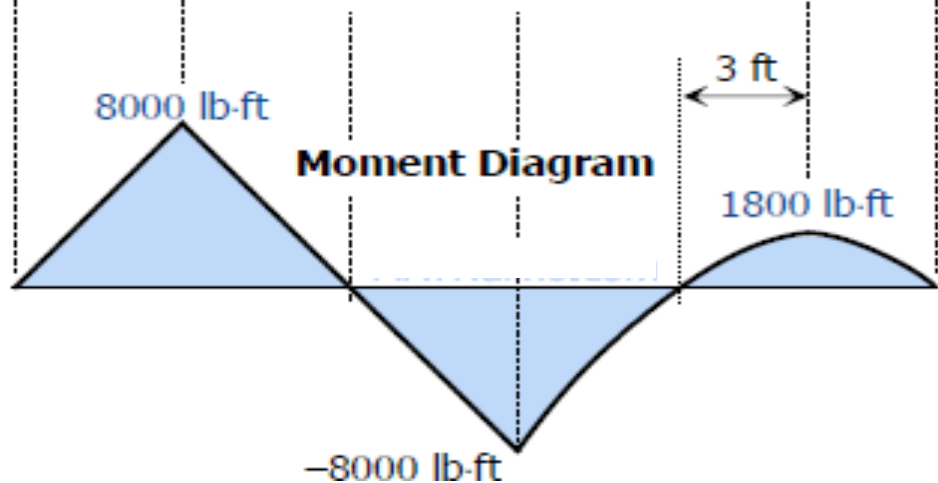




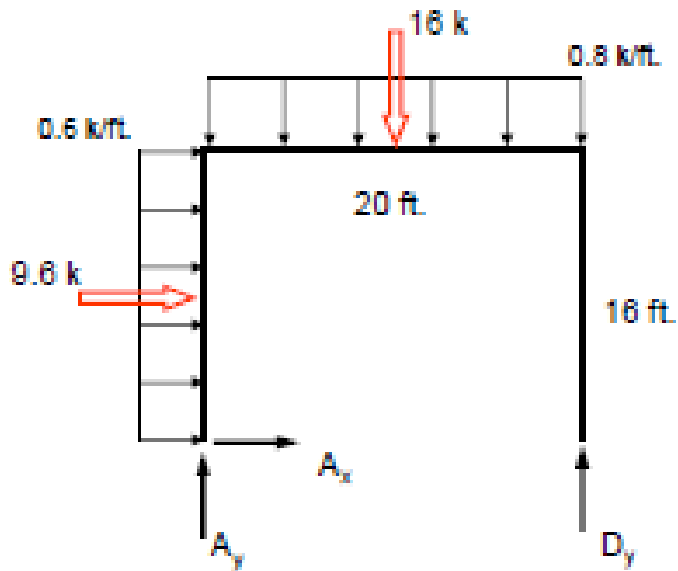
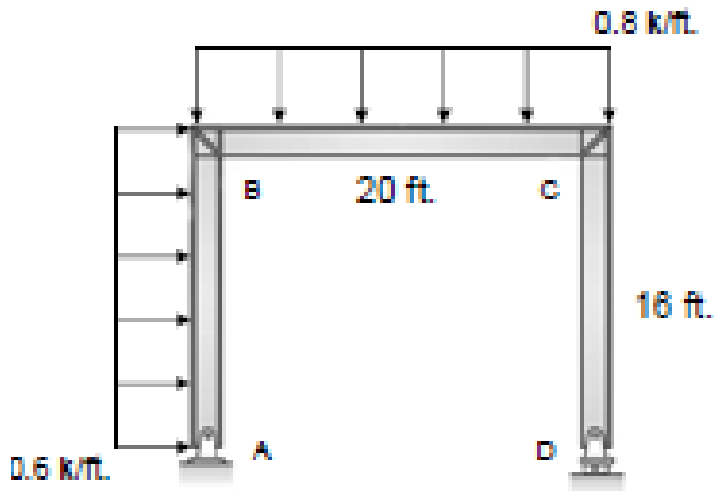
Load Diagram



Shear Diagram



Moment Diagram

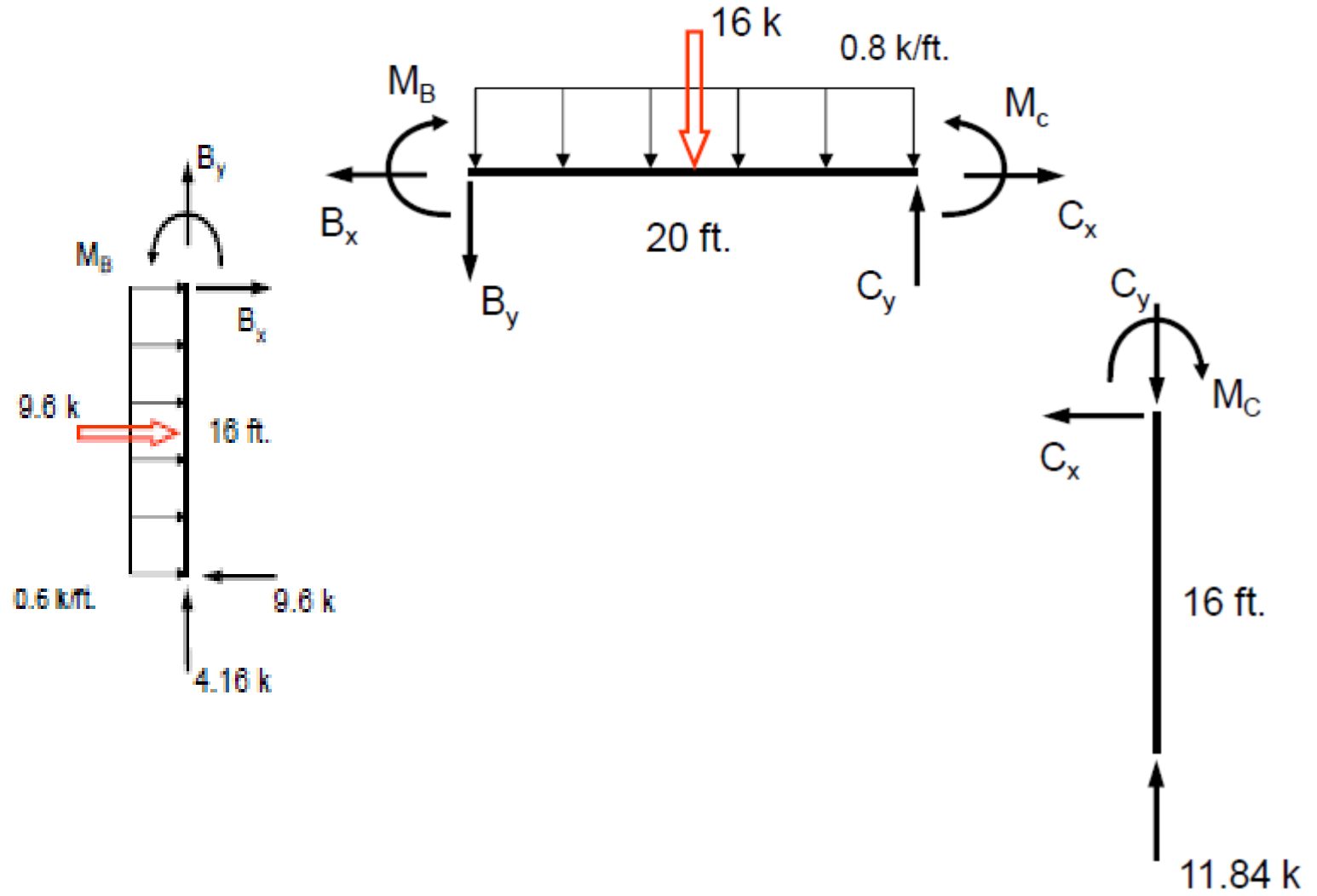


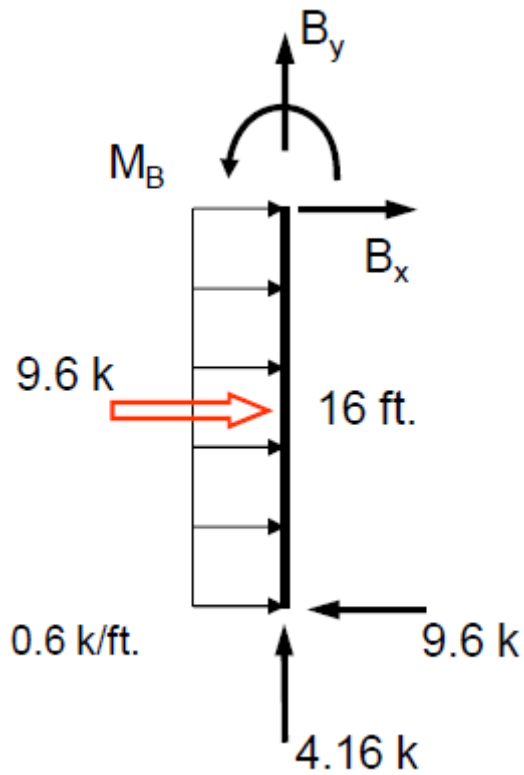
Free body diagram

$$\sum M_A = 0 ; D_y = 11.84 \text{ k}$$

$$A_y = 4.16 \text{ k}$$

$$A_x = -9.6 \text{ k}$$





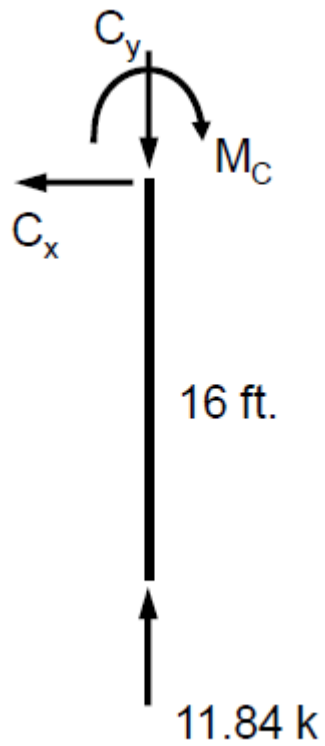
$$\sum M_B = 0 ; M_B = 76.8 \text{ k-ft}$$

$$B_y = -4.16 \text{ k}$$

$$B_x = 0 \text{ k}$$

$$V_y = 9.6 - 0.6 x$$

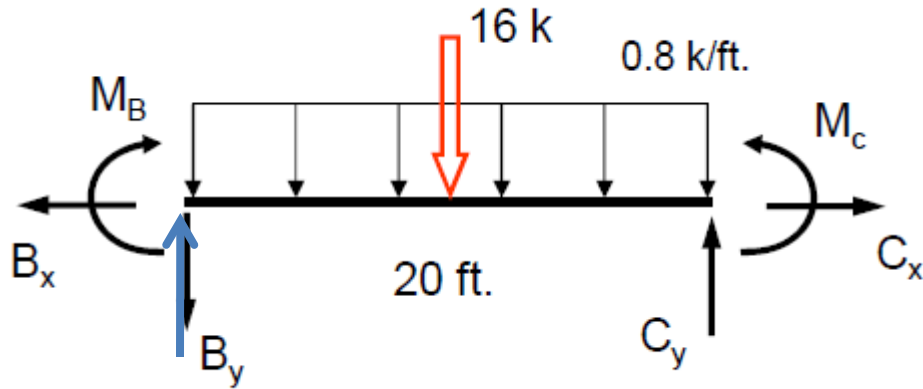
$$M_y = 9.6 * x - (0.6 x * x / 2)$$



$$\sum M_C = 0 = -M_C$$

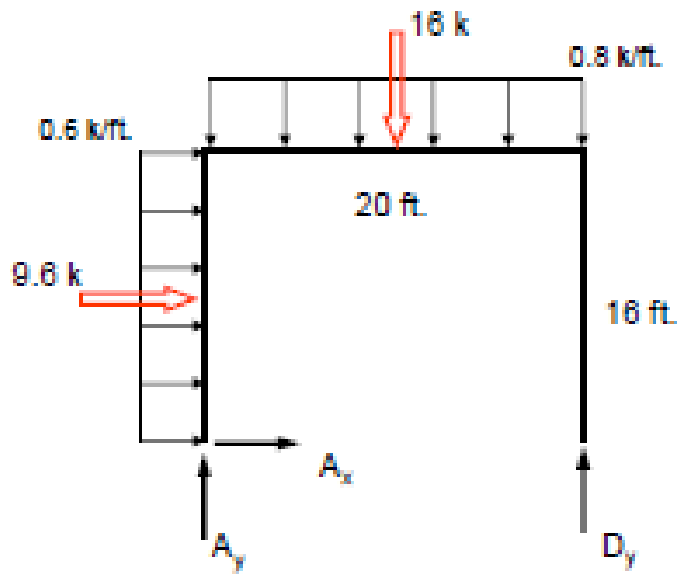
$$C_y = 11.84 \text{ k}$$

$$C_x = 0 \text{ k}$$

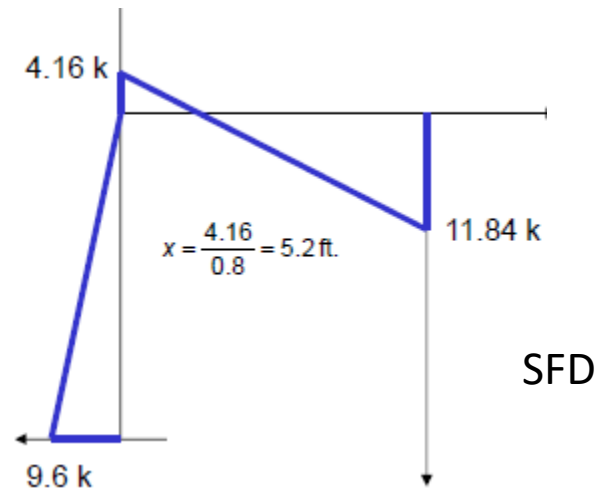


$$V_x = 4.16 - 0.8 x$$

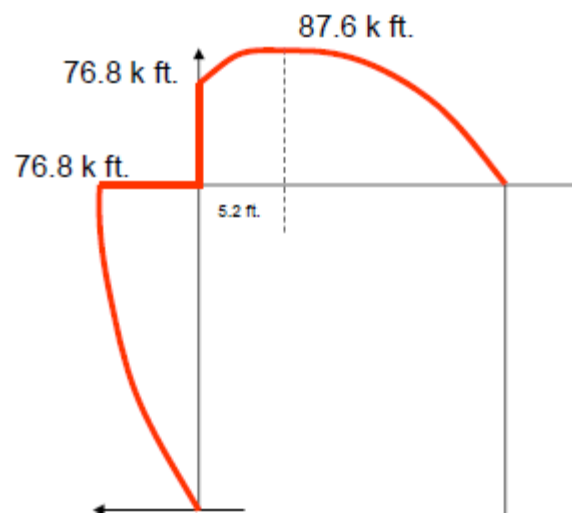
$$M_x = 4.16 * x + 76.8 - (0.8 x * x / 2)$$



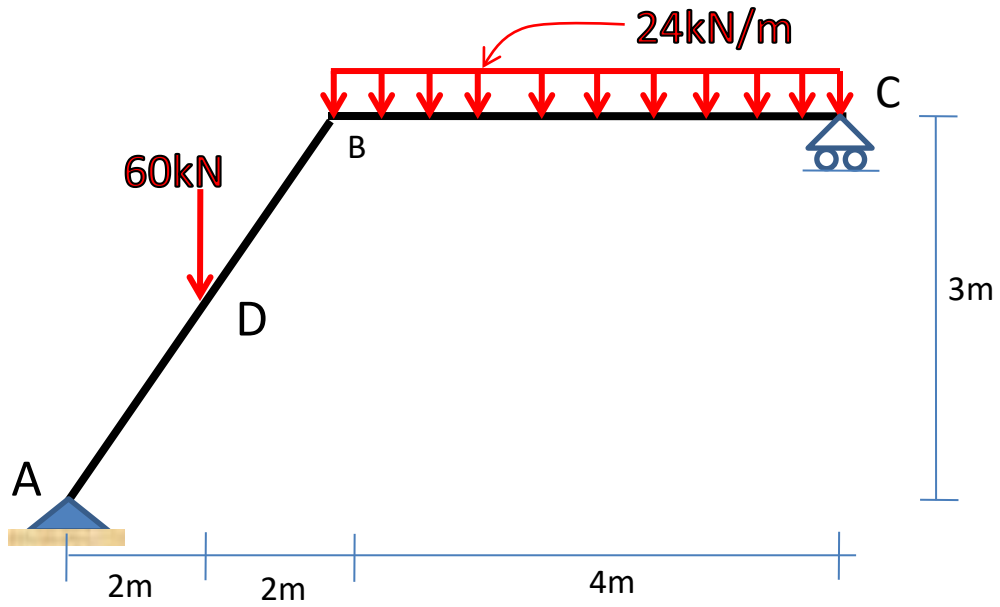
Free body diagram

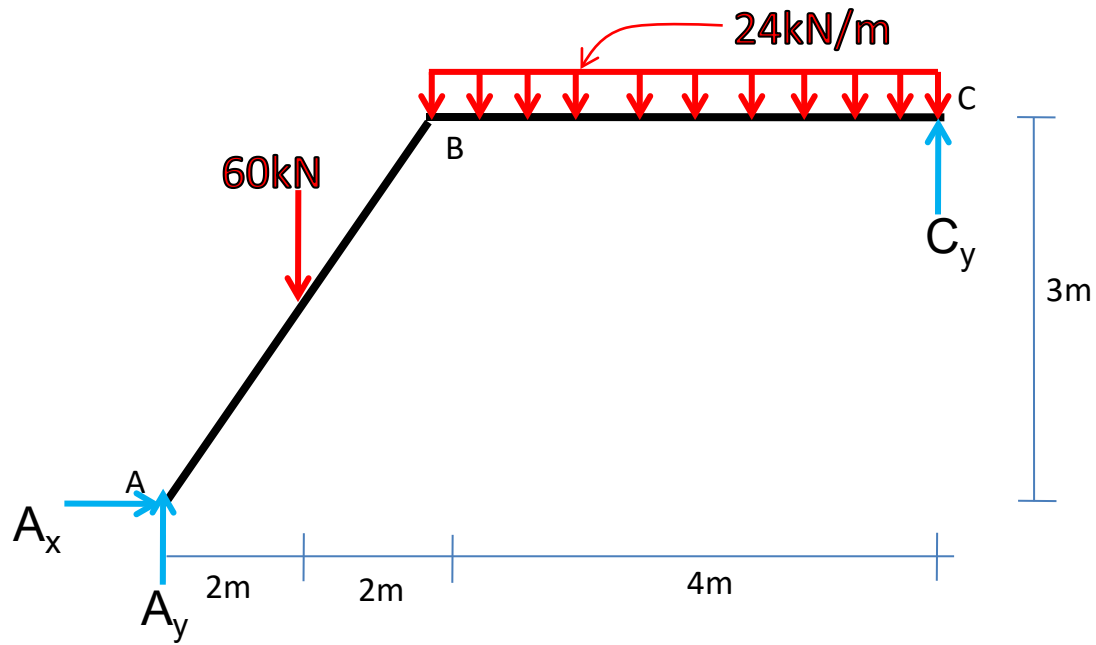


SFD



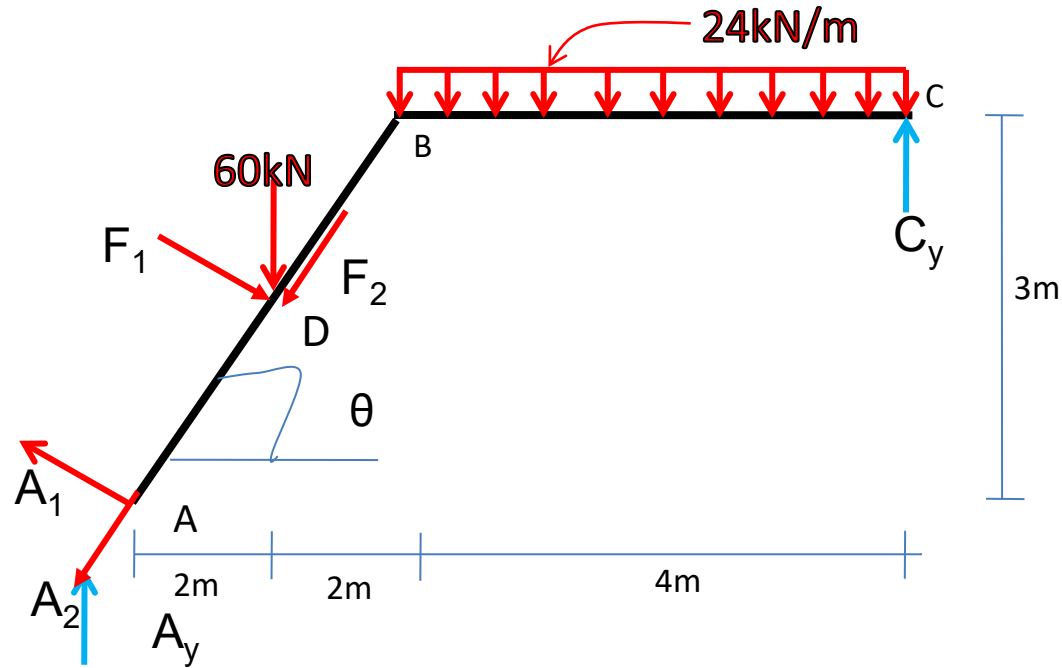
BMD





$$\sum M_A = 0; \quad 60 \cdot 2 + 24 \cdot 4 \cdot (4 + 4/2) - C_y \cdot 8 = 0 \quad C_y = 87 \text{ kN}$$

$$\sum F_y = 0; \quad A_y = 69 \text{ kN} \quad \sum F_x = 0; \quad A_x = 0 \text{ kN}$$



$$\cos \theta = 4/5$$

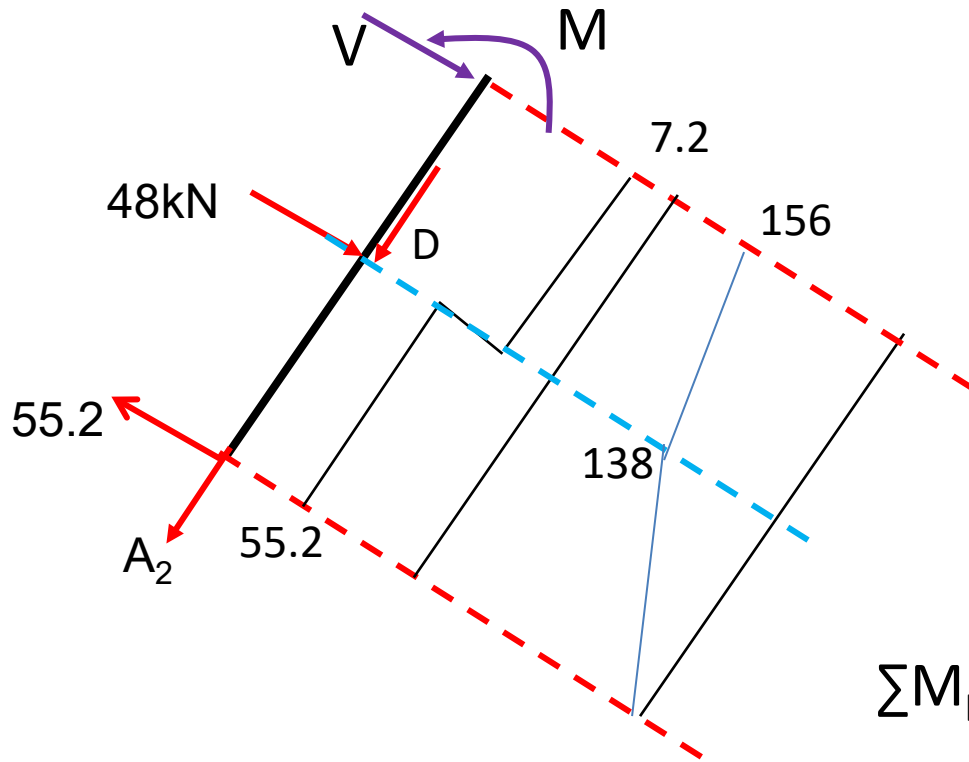
$$\sin \theta = 3/5$$

$$A_1 = A_y \cos \theta = 69 * 4/5 = 55.2 \text{ kN}$$

$$A_2 = -A_y \sin \theta = -69 * 3/5 = -41.4 \text{ kN}$$

$$F_1 = 60 \cos \theta = 60 * 4/5 = 48 \text{ kN}$$

$$F_2 = -60 \sin \theta = -60 * 3/5 = -36 \text{ kN}$$



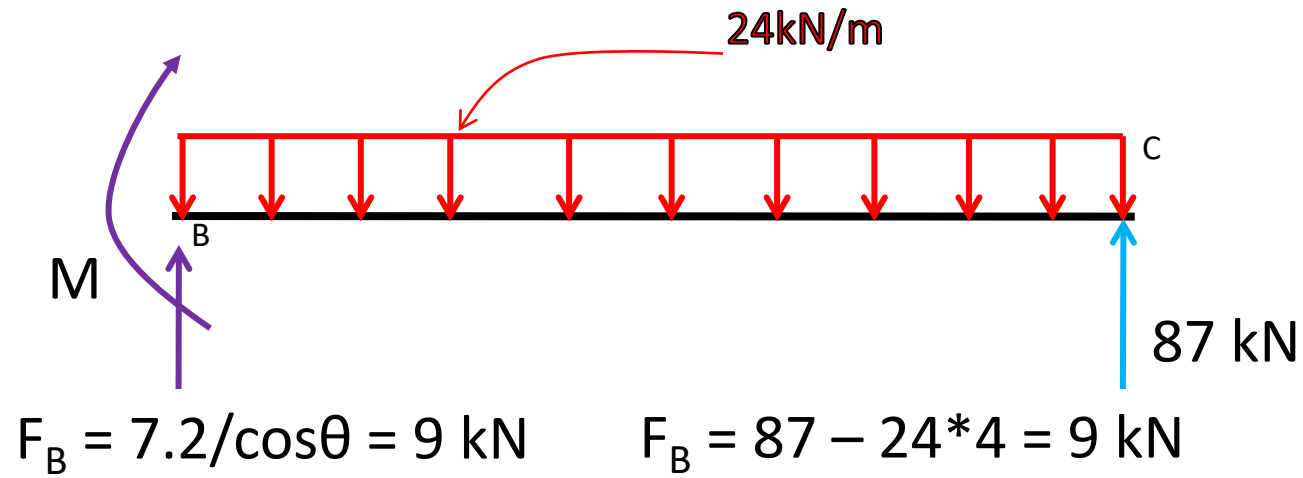
$$\sum M_B = 0; 55.2 * 2.5 - 48 * 2.5 - M = 0$$

$$M = 156 \text{ kN-m}$$

$$M_A = 0$$

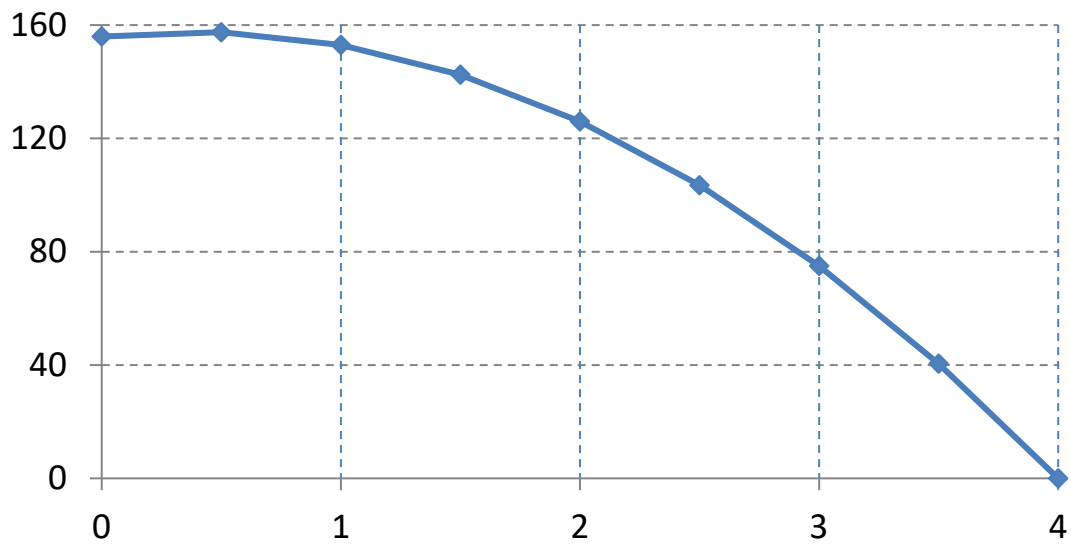
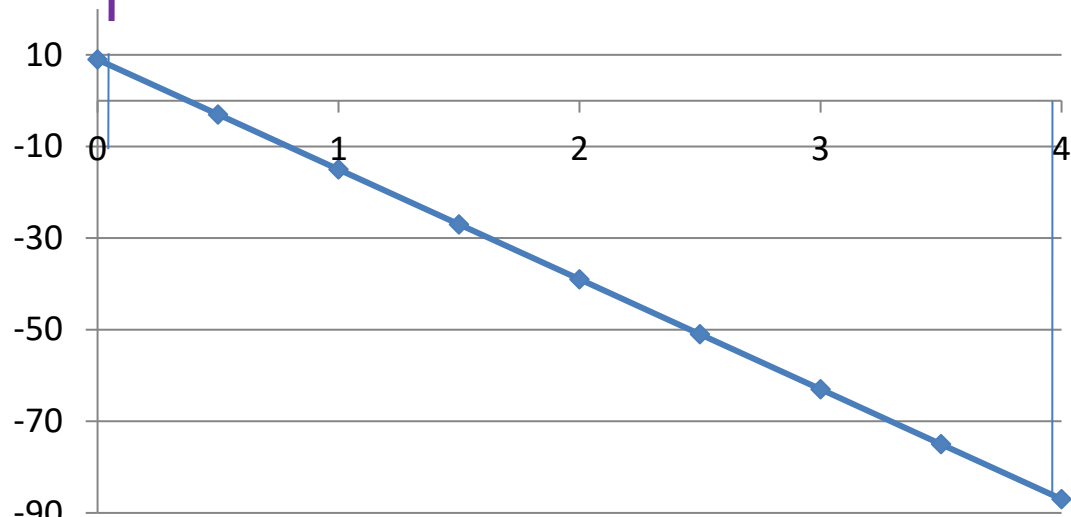
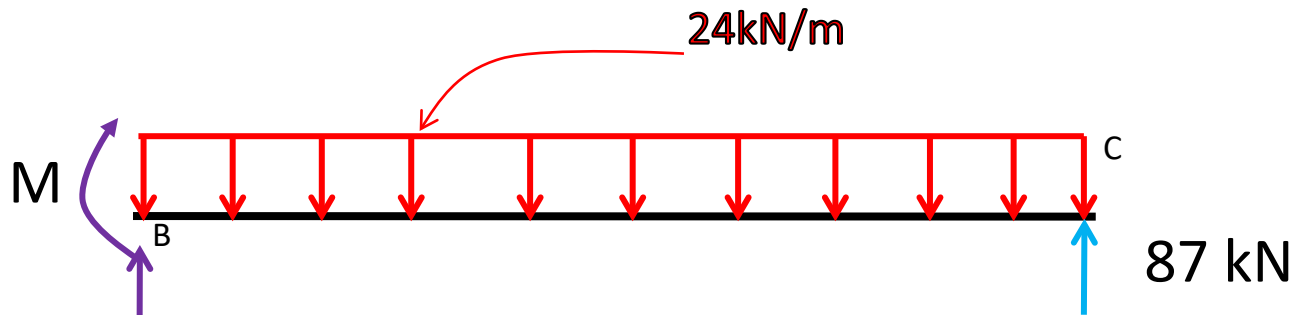
$$M_D = 55.2 * 2.5 = 138 \text{ kN-m}$$

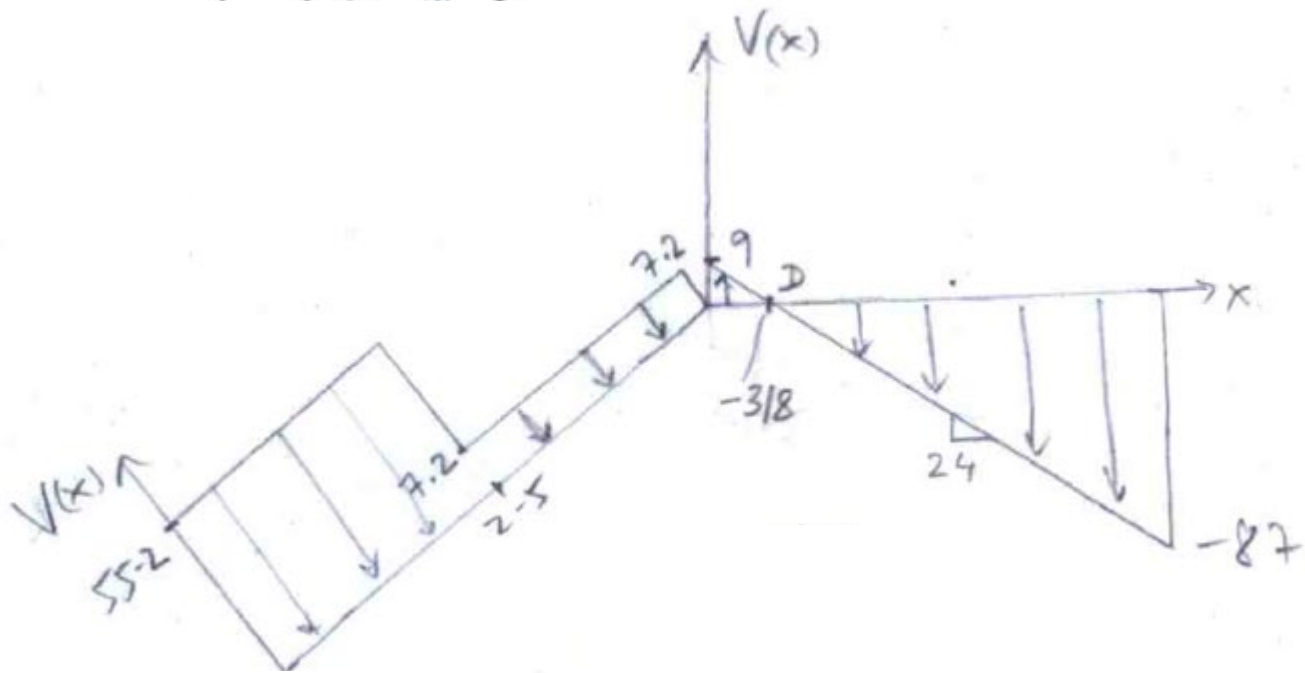
$$M_B = 55.2 * 5 - 48 * 2.5 = 156 \text{ kN-m}$$



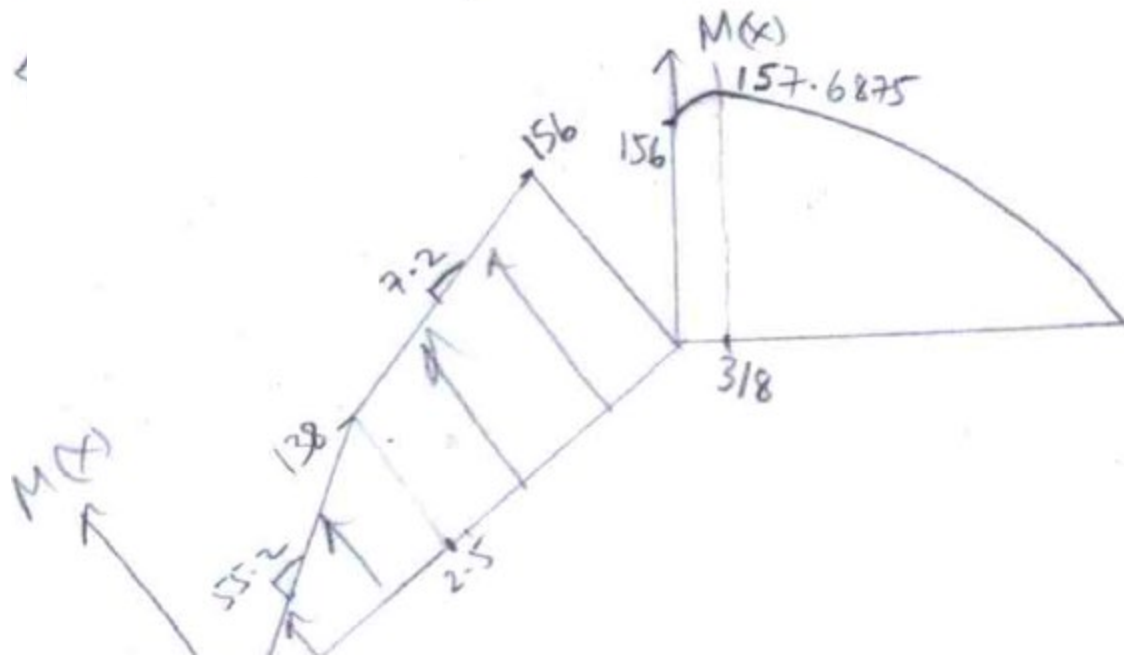
$$V_x = 9 - 24 * x$$

$$M_x = 156 + 9 * x - 24 * x * x / 2$$





SFD



BMD

